



Implementation and Application of a new Plasticity Model in LS-DYNA including Lode Angle Dependence

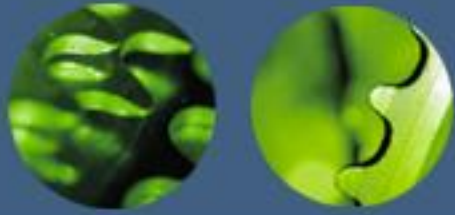
F.J.P.Reis

Department of Mechanical Engineering, Faculty of Engineering, University of Porto

F.X.C. Andrade

(DYNAmore GmbH)





OUTLINE

1. Motivation

- Definition of the Lode angle and normalized third invariant
- Materials with Lode angle dependent plasticity
- Existing plasticity models with Lode angle dependency

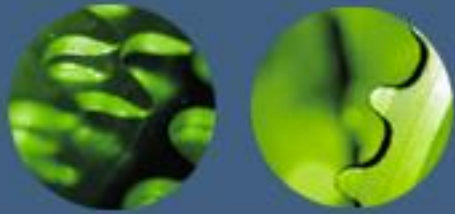
2. New plasticity model

- Base assumptions and desired features
- Constitutive equations: yield function and flow rule
- Graphic representation on the σ_1 - σ_2 space
- Numerical implementation

3. Numerical examples

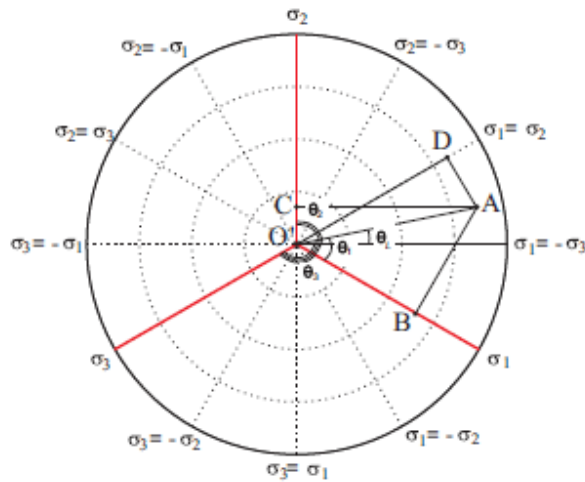
- Tensile stress states: notched and flat grooved specimens
- Shear stress states: butterfly specimen
- Compressive stress states: upset test

4. Final remarks



1. Motivation

Lode angle definition



Taken from "Xue, L. (2007), Ductile Fracture Modeling – Theory, Experimental Investigation, and Numerical Verification, PhD thesis, Massachusetts Institute of Technology, Cambridge, MA."

- Geometrically, the Lode Angle is the “smallest angle between the line of pure shear and the projection of the stress tensor on the deviatoric plane” (L.Malcher et al.)

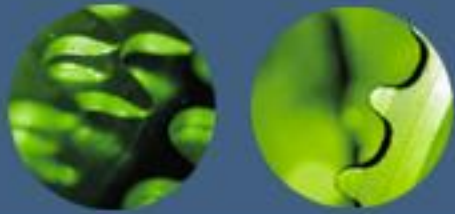
$$\theta_L = \tan^{-1} \left\{ \frac{1}{\sqrt{3}} \left[2 \left(\frac{s_2 - s_3}{s_1 - s_3} \right) - 1 \right] \right\}, \theta_L = \left[0, \frac{\pi}{3} \right]$$

- The Lode Angle θ_L is related with the normalized third deviatoric stress invariant, ξ

$$\xi = \cos(3\theta_L) = \frac{27 \det(\mathbf{s})}{2 \sigma_{eq}^3}, \xi = [-1, 1]$$

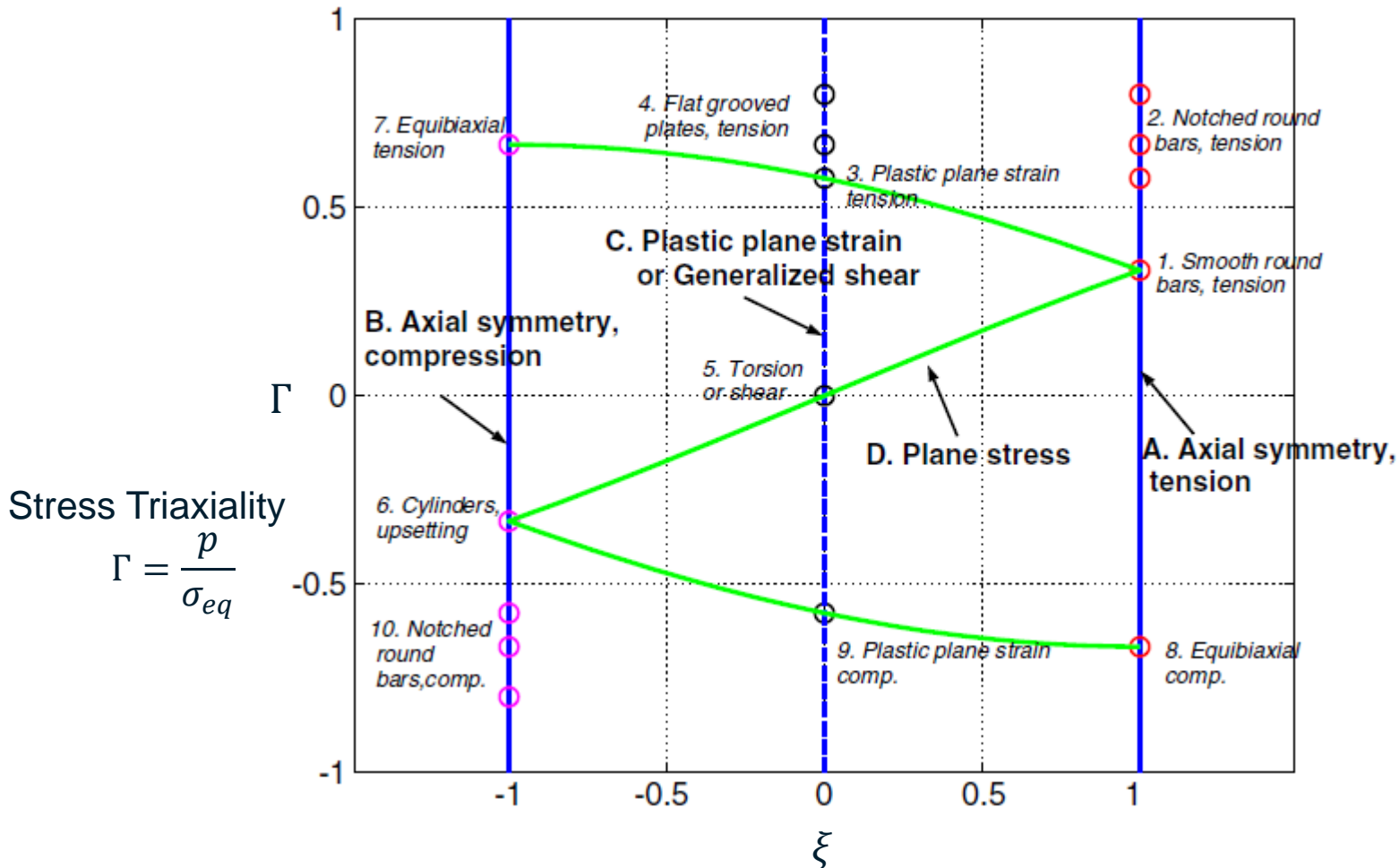
- Hereinafter, the normalized third deviatoric stress invariant, ξ , will be denoted as Lode Angle.

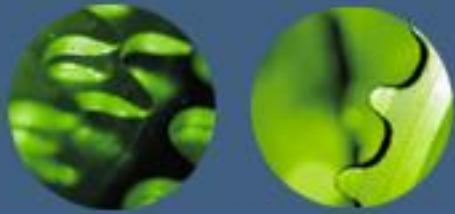
In LS-DYNA, the normalized third deviatoric stress invariant, ξ , is denoted as the “Lode Angle Parameter”



1. Motivation

Lode Angle Definition, ξ

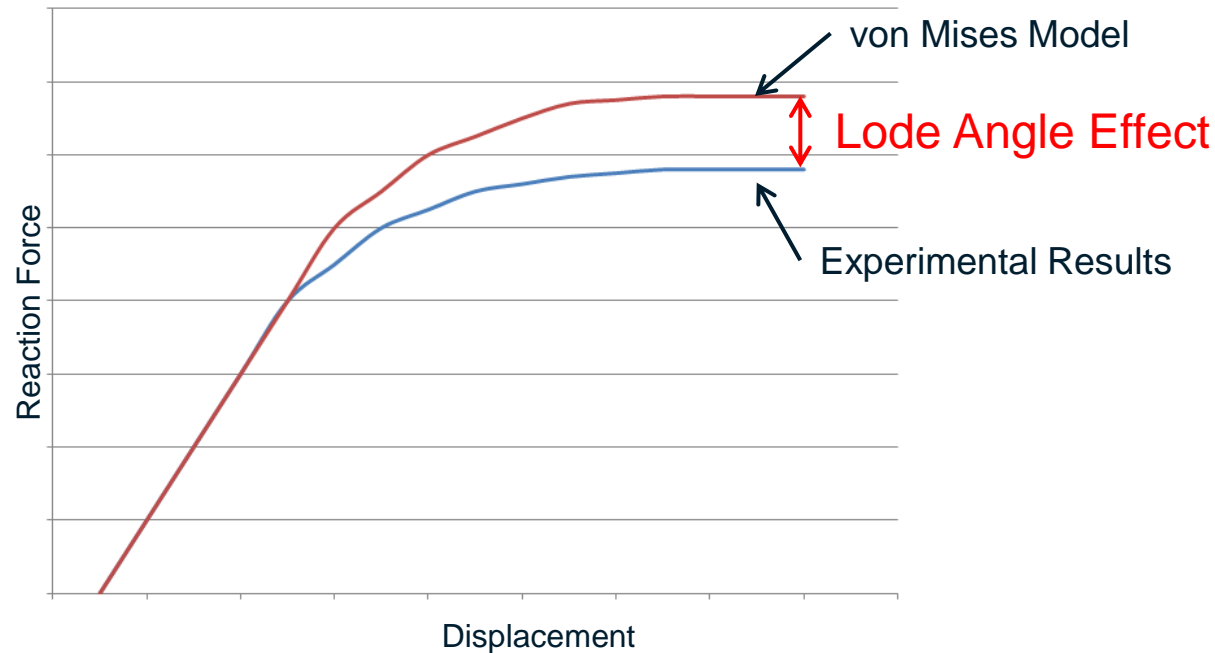
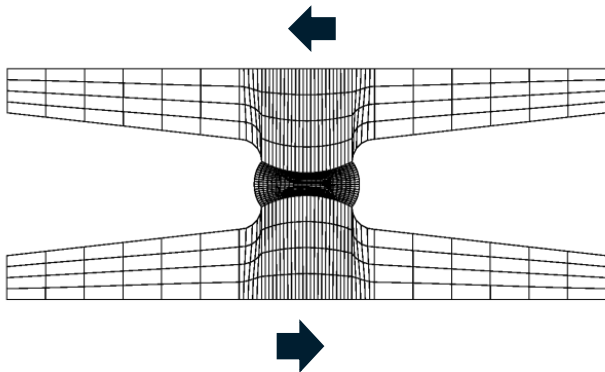


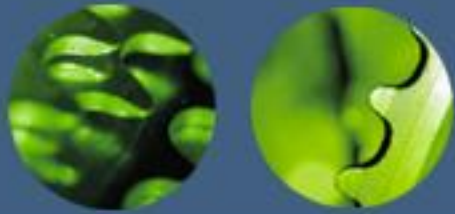


1. Motivation

Aluminium alloys

- Recent experimental analysis have proved that the Lode Angle have a considerable effect on the stress-plastic relation of some aluminium alloys (Y. Bai et al., Mirone et al, X. Gao et al.)
- In contrast, they also concluded that the hydrostatic pressure has a negligible effect





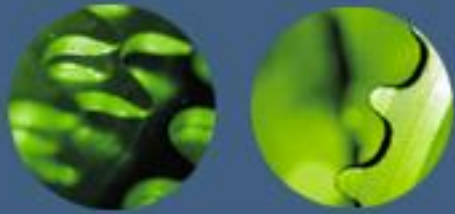
1. Motivation

Aluminium alloys

- However, the effect of the Lode Angle is not constant.
- It depends on the stress state!

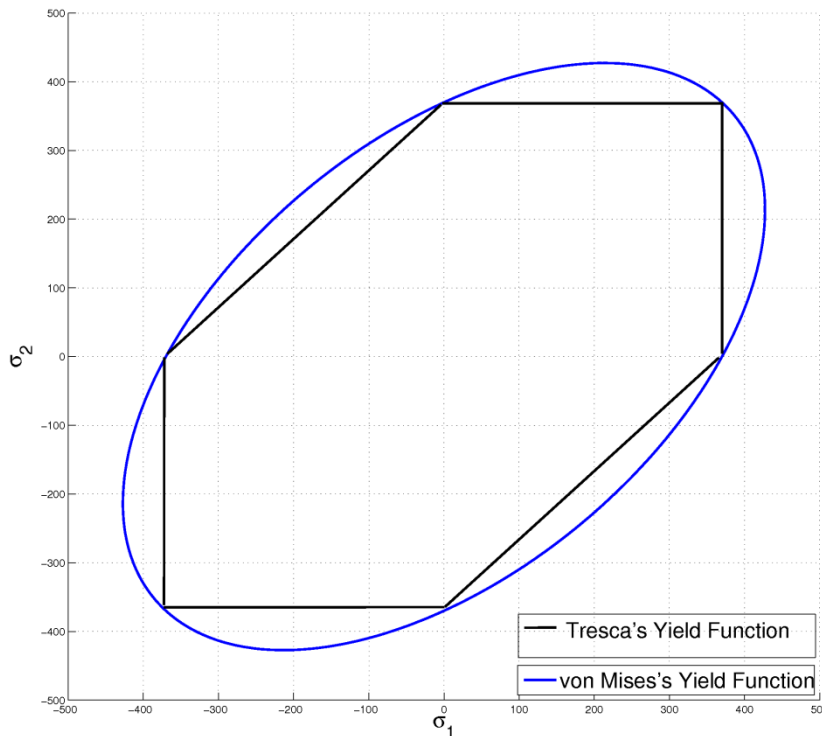
Stress State/Specimen	ξ	Lode Angle Effect (according to experimental evidences)
1. Tensile round bars	$\xi = 1$	N
2. Compression round bars	$\xi = -1$	N
3. Shear stress states	$\xi = 0$	Y
4. Plane Strain Specimen - Traction	$\xi = 0$	Y
5. Plane Strain Specimen – Compression	$\xi = 0$	Y
5. Bi-axial tension	$\xi = 1$	N
6. Bi-axial compression	$\xi = -1$	N

The maximum Lode Angle effect takes place when $\xi = 0$



1. Motivation

Tresca's yield function



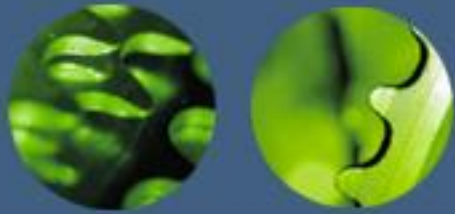
$$\Phi = (\sigma_{max} - \sigma_{min}) - \sigma_y$$

Advantages:

- It is a classical, well-established yield function

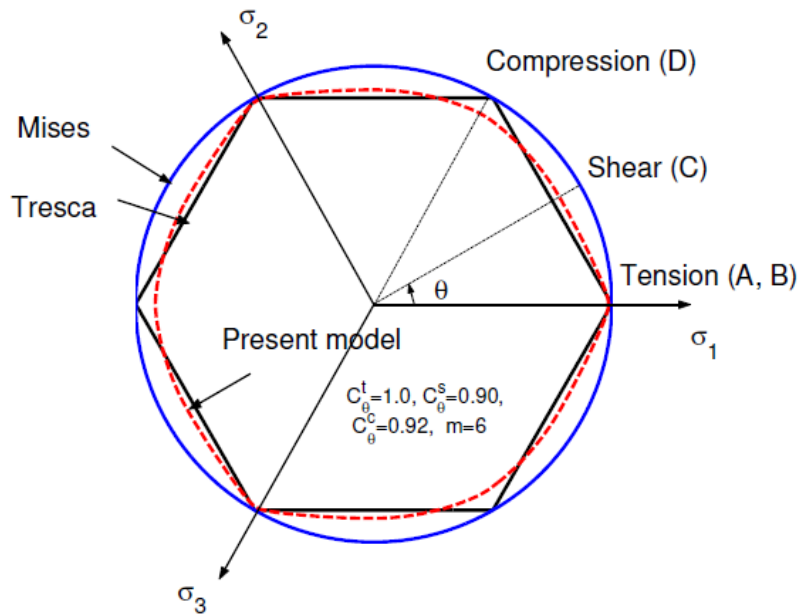
Disadvantages:

- The effects of the 3rd invariant are fixed
- The yield function is non-continuous, making its numerical implementation more difficult (directional derivatives are needed!)



1. Motivation

Bai & Wierzbicki's yield function



Taken from "Bai, Y. (2008), Effect of loading history on necking and fracture, PhD thesis, Massachusetts Institute of Technology, Cambridge, MA."

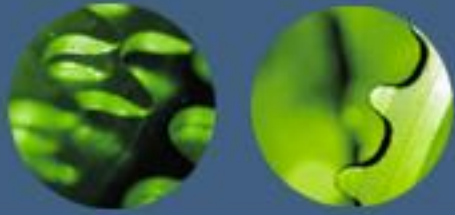
$$\Phi = \sigma_{eq} - \sigma_y [1 - c_\Gamma (\Gamma - \Gamma_0)] \left[c_\theta^s \left(\phi - \frac{\phi^{m+1}}{m+1} \right) + (c_\theta^{ax} - c_\theta^s) \left(\phi - \frac{\phi^{m+1}}{m+1} \right) \right]$$

Advantages:

- The convexity of the yield function has to be ensured by a numerical parameter

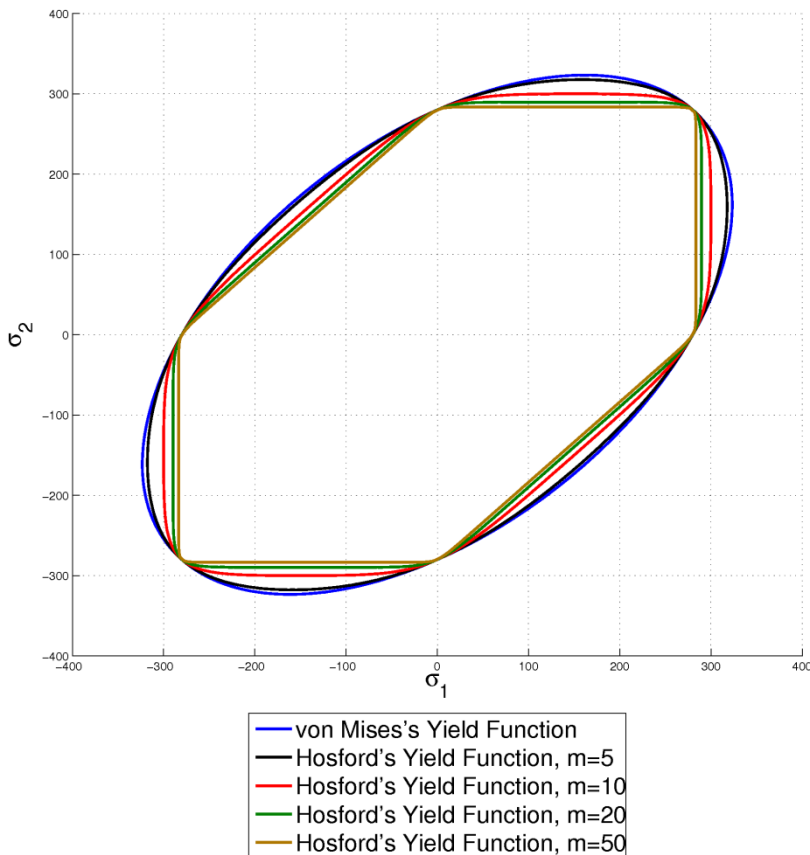
Disadvantages:

- 7 material parameters plus a hardening curve are required for calibration
- Isochoric plasticity is questionable
- Complex to implement



1. Motivation

Hosford's yield function



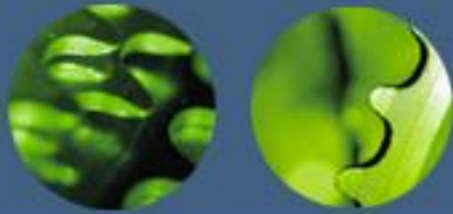
$$\Phi = |\sigma_1 - \sigma_2|^m + |\sigma_2 - \sigma_3|^m + |\sigma_3 - \sigma_1|^m - 2\sigma_y^m$$

Advantages:

- It is available in LS-DYNA through MAT_36 if parameters are set to be isotropic

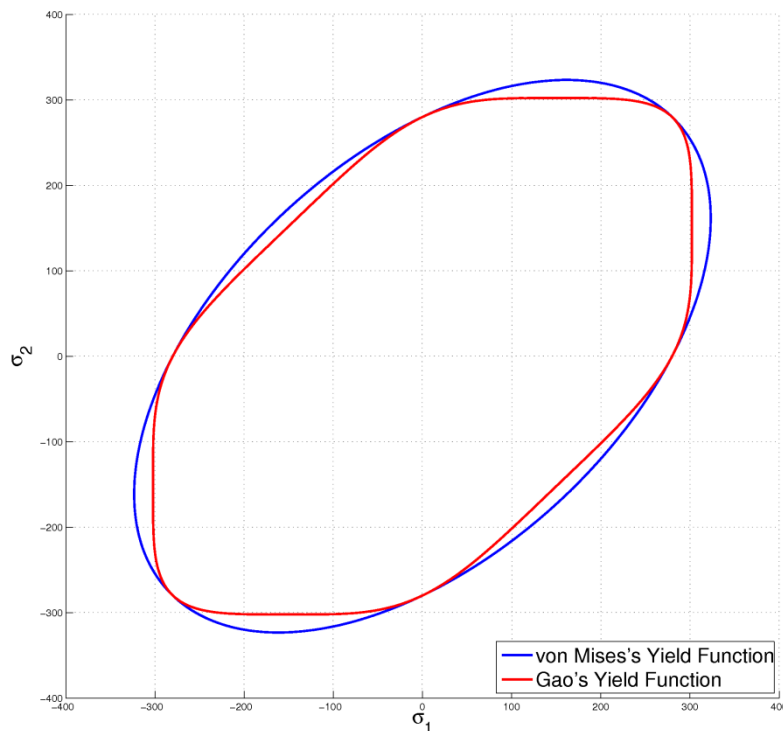
Disadvantages:

- Effect of 3rd invariant can be controlled by the exponent “m”, but the physical meaning of “m” is somewhat difficult to grasp
- Formulated in principal stress space



1. Motivation

Gao's yield function



$$a_1 = 0, b_1 = 60.75 \text{ and } c_1 = 1.07$$

$$\Phi = c_1 (a_1 I_1^6 + 27 J_2^3 + b_1 J_3^2)^{1/6} - \sigma_y$$

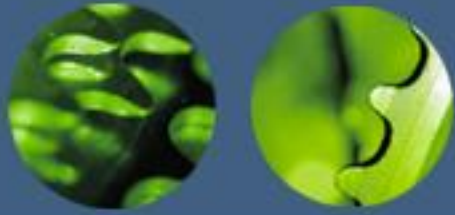
a_1 and b_1 are material parameters and c_1 is a function of a_1 and b_1

Advantages:

- It is possible to control the effect of the third invariant

Disadvantages:

- In addition to the hardening curve, 6 more material parameters are required which have no physical meaning
- Non-associated model
- The yield function may be non-convex!



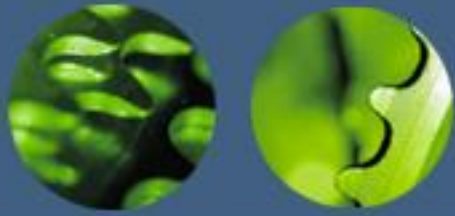
2. New Plasticity Model

Desired features

- Starting from classical J_2 von Mises plasticity (*MAT_24)
- J_2 plasticity should be easily recovered if material is not dependent on ξ
- Easiness of calibration (as less new parameters as possible)
- Dependency of ξ should be simple to grasp

Main assumptions for the new model

- Material has a different yield stress under tensile and shear stress states
- The dependency of the yield stress in respect to ξ is assumed quadratic



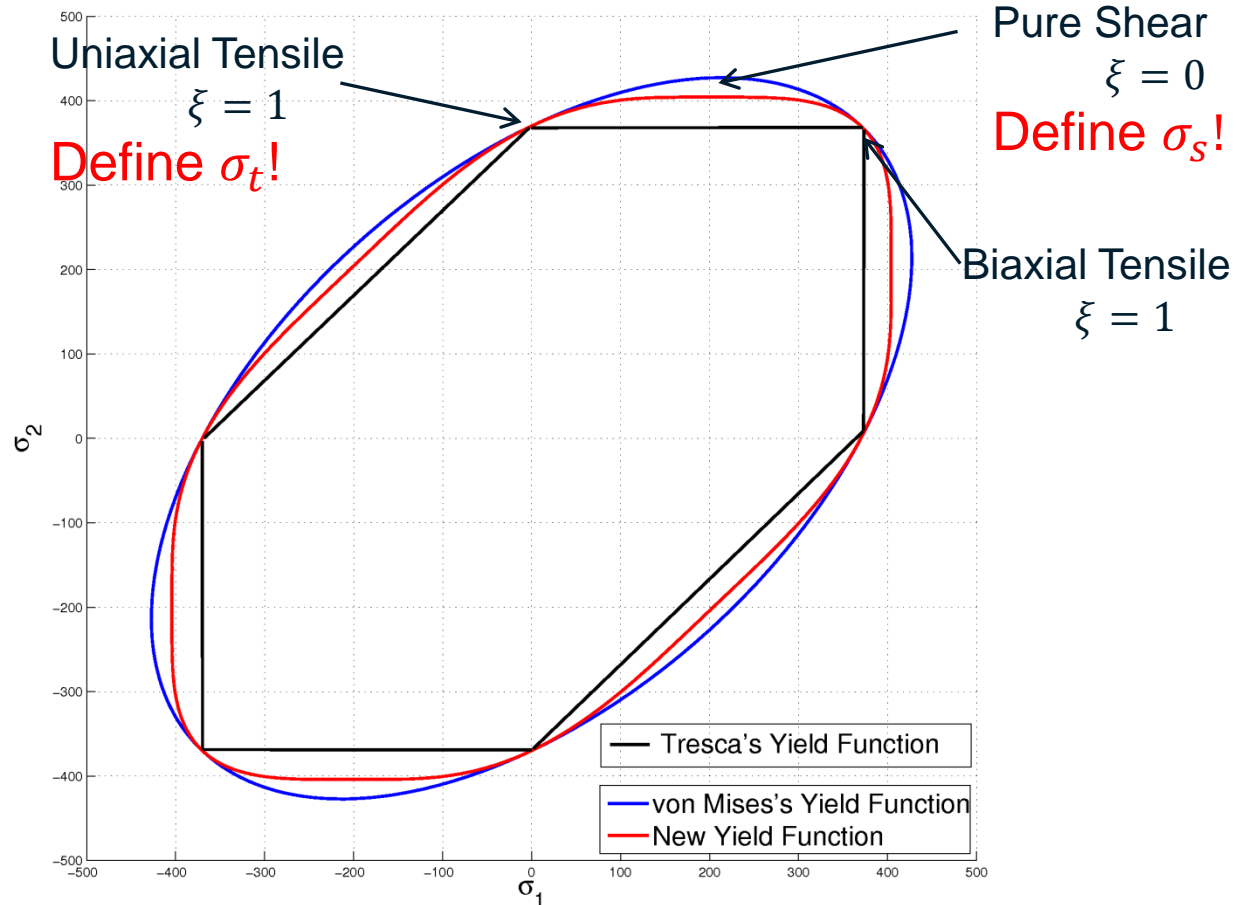
2. New Plasticity Model

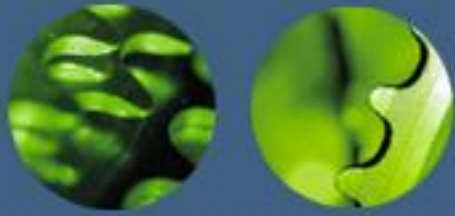
Proposed yield function

$$\Phi = \sigma_{eq} + (\sigma_t - \sigma_s)(1 - \xi^2) - \sigma_y$$

Parameters definition:

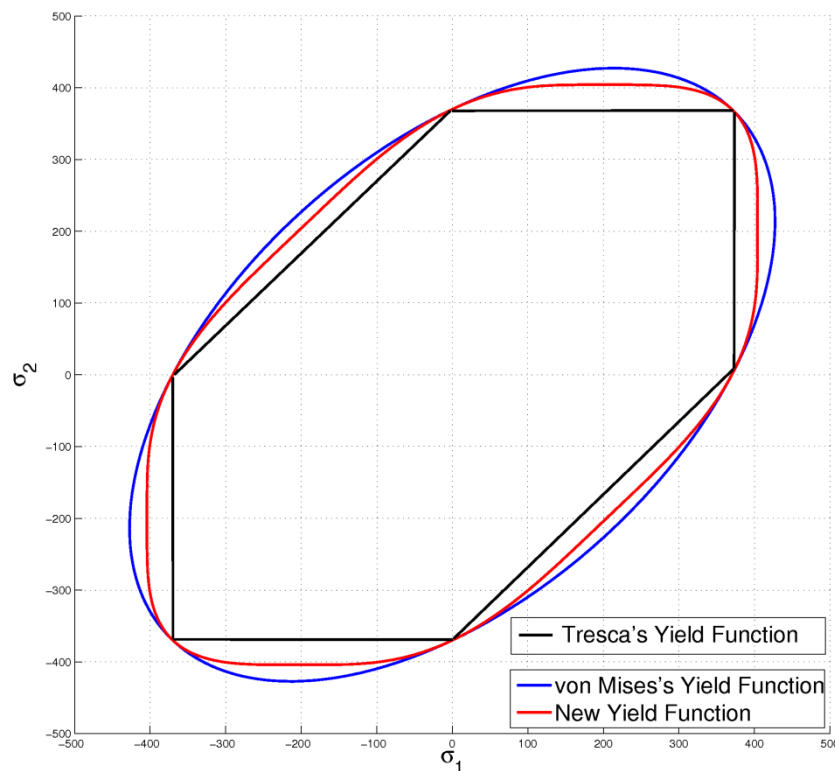
1. Hardening Curve:
The same as used in von Mises Model
2. Tensile Yield Stress, σ_t . Determined through standard tensile test
3. Shear Yield Stress, σ_s . Determined through a shear test





2. New Plasticity Model

Proposed yield function



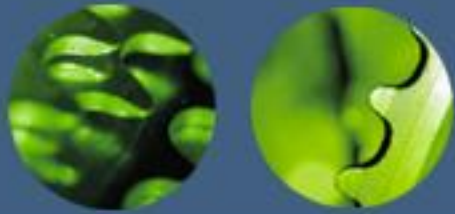
$$\Phi = \sigma_{eq} + (\sigma_t - \sigma_s)(1 - \xi^2) - \sigma_y$$

Advantages:

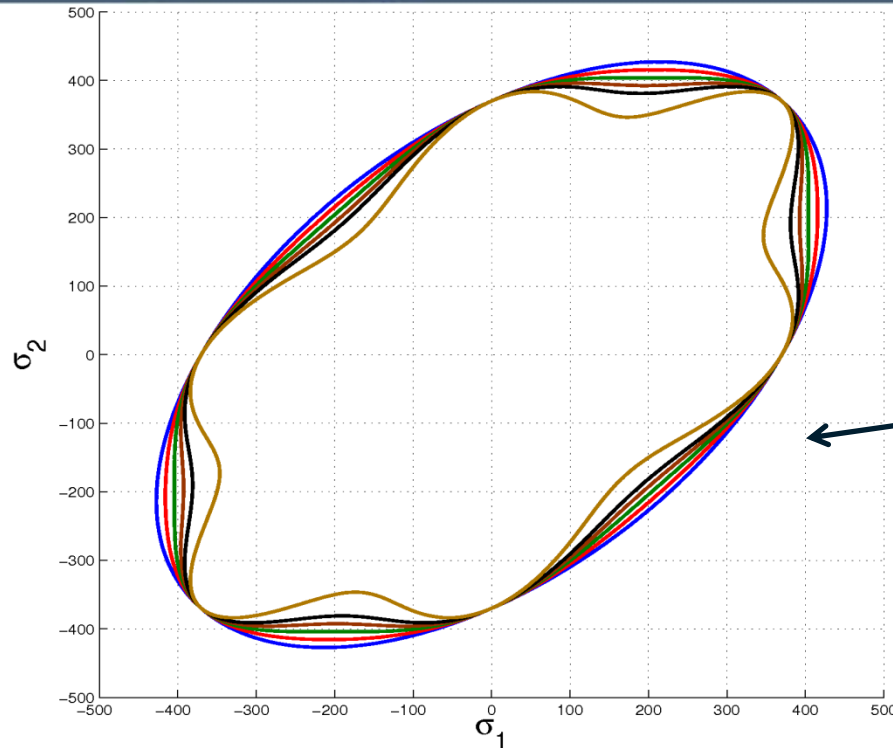
- Only one extra parameter in comparison to classical J_2 plasticity (*MAT_24)
- ξ -dependency is easier to grasp than other plasticity models
- Only two physical tests are required for calibration: tensile and shear test
- The yield function is always continuous
- The model is relatively simple to implement

Disadvantage:

- Yield surface may be non-convex!



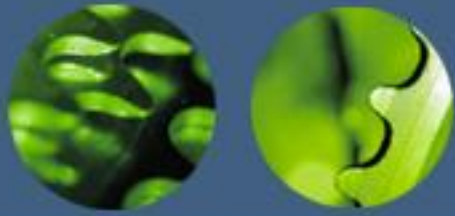
2. New Plasticity Model



Non-convexity may arise for large differences between σ_t and σ_s !

$$\Phi = \sigma_{eq} + (\sigma_t - \sigma_s)(1 - \xi^2) - \sigma_y$$

- von Mises's Yield Function
- New Yield Function, $\sigma_t=370$, $\sigma_s=360$
- New Yield Function, $\sigma_t=370$, $\sigma_s=350$
- New Yield Function, $\sigma_t=370$, $\sigma_s=340$
- New Yield Function, $\sigma_t=370$, $\sigma_s=330$
- New Yield Function, $\sigma_t=370$, $\sigma_s=300$



2. New Constitutive Model: Constitutive relations

Yield function

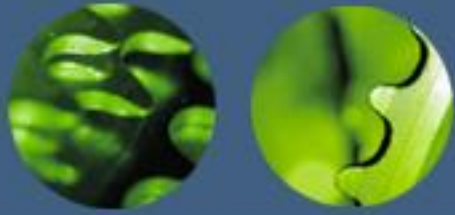
$$\Phi = \sigma_{eq} + (\sigma_t - \sigma_s) \cdot (1 - \xi^2) - \sigma_y = 0$$

Associative plastic flow

$$\dot{\boldsymbol{\varepsilon}}^p = \dot{\gamma} \frac{\partial \Phi}{\partial \boldsymbol{\sigma}} = \left[\sqrt{\frac{3}{2}} \frac{\mathbf{s}}{\|\mathbf{s}\|} - 2 \cdot (\sigma_t - \sigma_s) \xi \frac{\partial \xi}{\partial \boldsymbol{\sigma}} \right]$$

Plastic work equivalence

$$\sigma_{eq} \cdot \bar{\boldsymbol{\varepsilon}}^p = \boldsymbol{\sigma} : \dot{\boldsymbol{\varepsilon}}^p$$



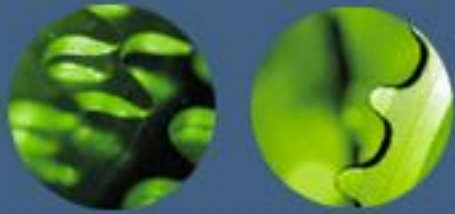
2. New Constitutive Model: Numerical implementation

The following system of equations has to be solved:

$$\left\{ \begin{array}{l} res_s = \mathbf{s}_{n+1} - 2G\boldsymbol{\varepsilon}_{n+1}^{e\,Trial} + 2G\Delta\gamma \left[\sqrt{\frac{3}{2} \frac{\mathbf{s}_{n+1}}{\|\mathbf{s}_{n+1}\|}} - 2 \cdot (\sigma_t - \sigma_s)\xi_{n+1} \frac{\partial \xi_{n+1}}{\partial \boldsymbol{\sigma}_{n+1}} \right] \\ res_{\varepsilon^p} = \bar{\varepsilon}_{n+1}^p - \bar{\varepsilon}_n^p - \frac{\boldsymbol{\sigma}_{n+1} : \Delta \boldsymbol{\varepsilon}^p}{\sigma_y(\bar{\varepsilon}_{n+1}^p)} \\ res_{\Delta\gamma} = \sigma_{eq_{n+1}} + (\sigma_t - \sigma_s) \cdot (1 - \xi_{n+1}^2) - \sigma_y(\bar{\varepsilon}_{n+1}^p) \end{array} \right.$$

The consistent tangent operator (implicit analysis) reads:

$$\mathbf{D}^{ep} = \frac{\partial \boldsymbol{\sigma}_{n+1}}{\partial \boldsymbol{\varepsilon}_{n+1}^{e\,Trial}} = 2G[\mathbf{A}]^{-1} : \left(\mathbf{I}_4 - \frac{1}{3} \mathbf{I}_2 \otimes \mathbf{I}_2 \right) + K \mathbf{I}_2 \otimes \mathbf{I}_2$$



3. Numerical Results

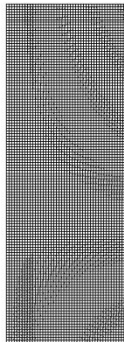
Material Properties

Young Modulus	$E = 71.15 \text{ GPa}$
Poisson's Ration	$\nu = 0.3$
Tensile yield stress	$\sigma_t = 370 \text{ MPa}$

Tensile Stress States

Smooth bar

(Specimen used to calibrate the hardening curve)



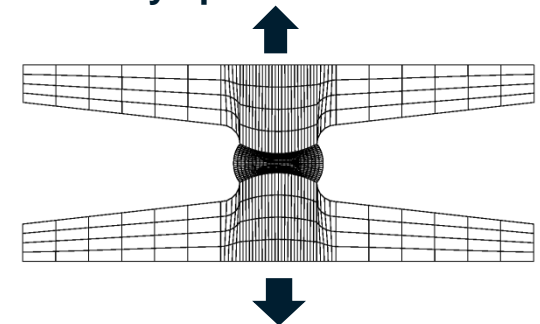
- Only a quarter was modelled
- 5760 Axisymmetric quadratic elements
- $D = 9\text{mm}$
- 5% reduction of section to trigger localization

Flat grooved specimen

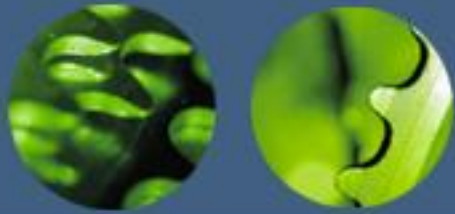


- Only a quarter was modelled
- 420 Axisymmetric quadratic elements
- Displacement Prescribed, $u = 0.4\text{mm}$
- $D = 5\text{mm}$, $R_{notched} = 1.59\text{mm}$, $l = 25\text{mm}$, $h = 50\text{mm}$

Butterfly specimen



- 3392 quadratic Hexahedral elements
- Displacement Prescribed, $u = 1\text{mm}$
- Geometry taken from "Bai, Y. (2008), Effect of loading history on necking and fracture, PhD thesis, Massachusetts Institute Technology, Cambridge, MA."



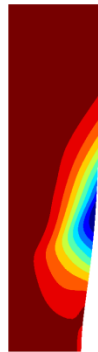
3. Numerical Results

Tensile Stress States: Smooth Bar

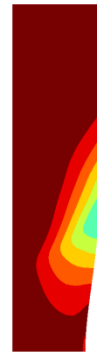
Von Mises

New Model, $\sigma_s = 340$

New Model, $\sigma_s = 325$



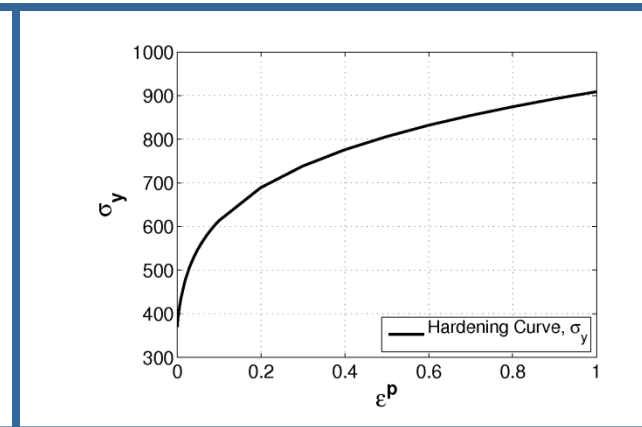
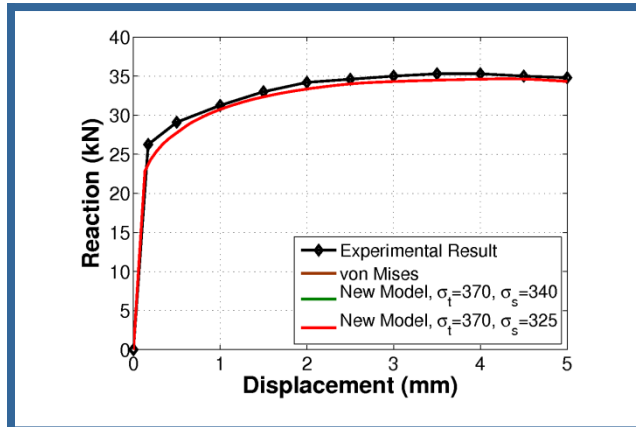
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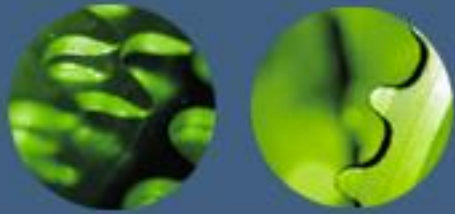
XI



XI

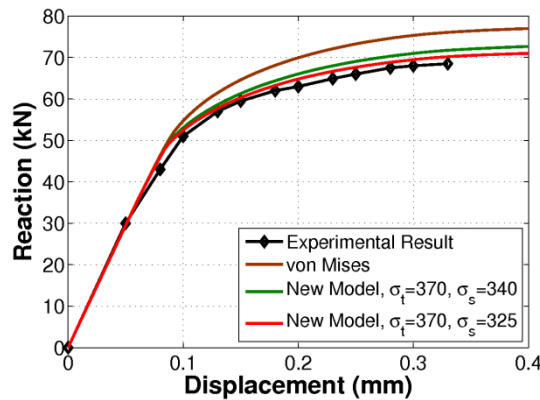


Hardening curve suggested by Bai et al.



3. Numerical Results

Tensile Stress States: Flat Grooved, $R = 1.59mm$

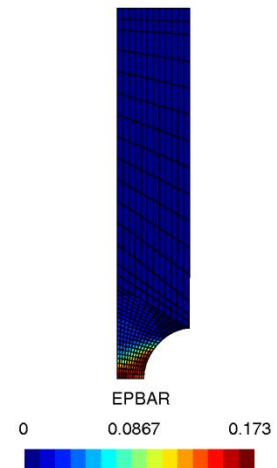
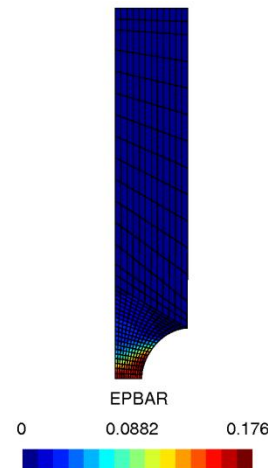
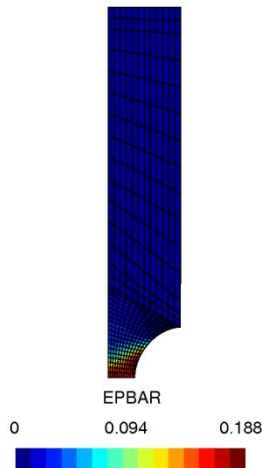


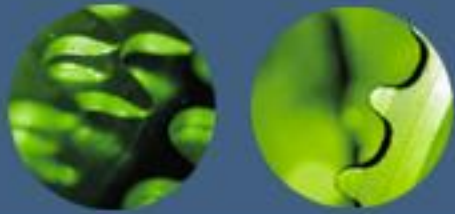
- The new model has the ability to properly capture the effects of ξ
- The appropriate value of $\sigma_s = 325 MPa$
- Despite the slight non-convexity of the yield function for $\sigma_s = 325 MPa$, no convergence problems were found \rightarrow In fact, the convergence rate is practically quadratic

Von Mises

New Model, $\sigma_s = 340$

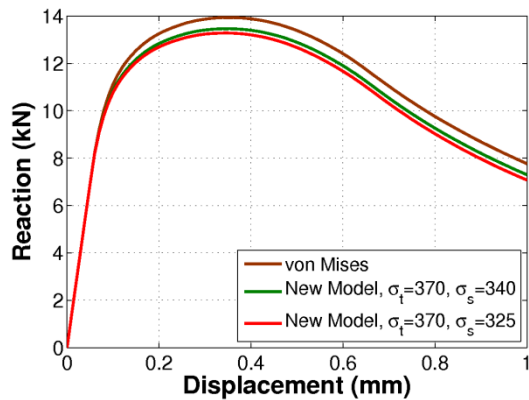
New Model, $\sigma_s = 325$





3. Numerical Results

Tensile Stress States: Butterfly Specimen

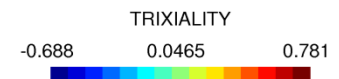
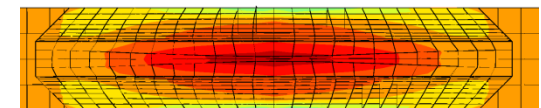
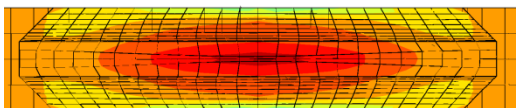
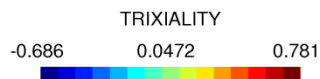
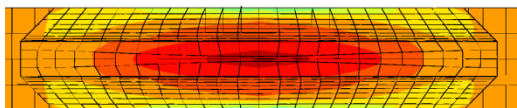
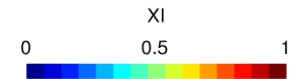
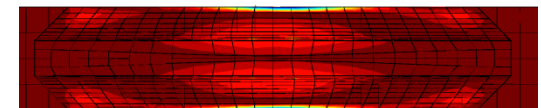
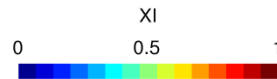
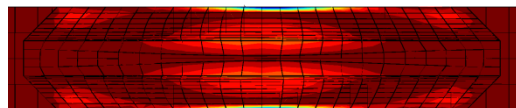
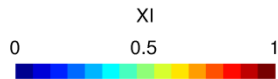
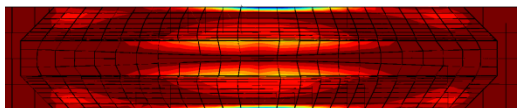


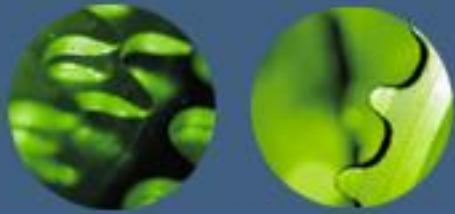
- The difference among the results is due to the fact that at the critical section ξ is not equal to one
- Shear effects associated with average stress triaxialities take place at the critical section
- This is a consequence of the geometry

Von Mises

New Model, $\sigma_s = 340$

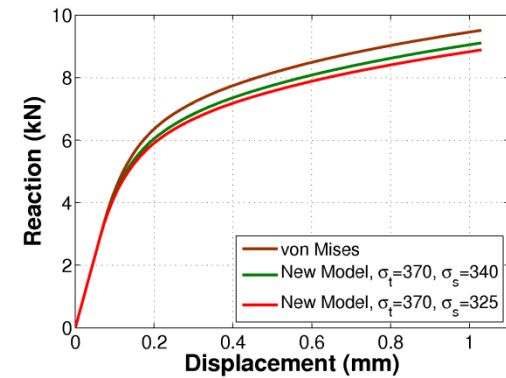
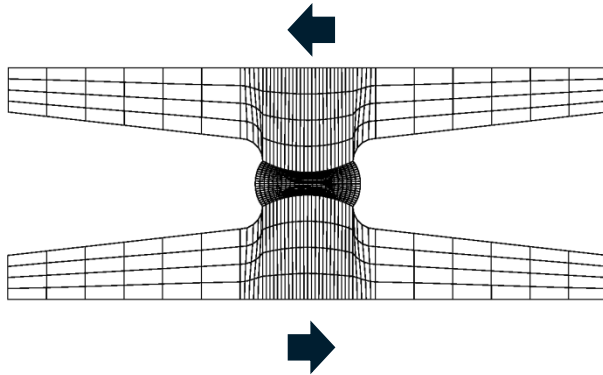
New Model, $\sigma_s = 325$





3. Numerical Results

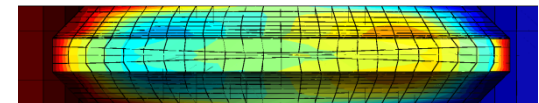
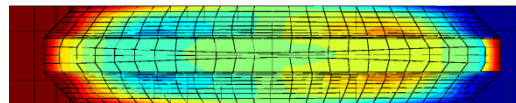
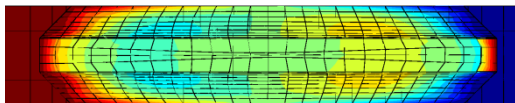
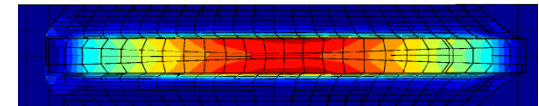
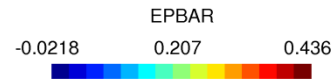
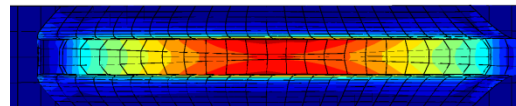
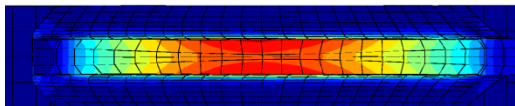
Shear Stress States: Butterfly Specimen

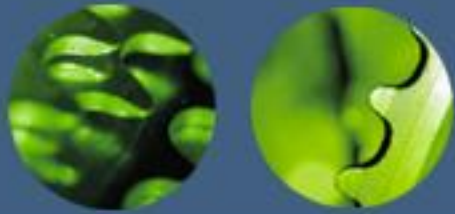


Von Mises

New Model, $\sigma_s = 340$

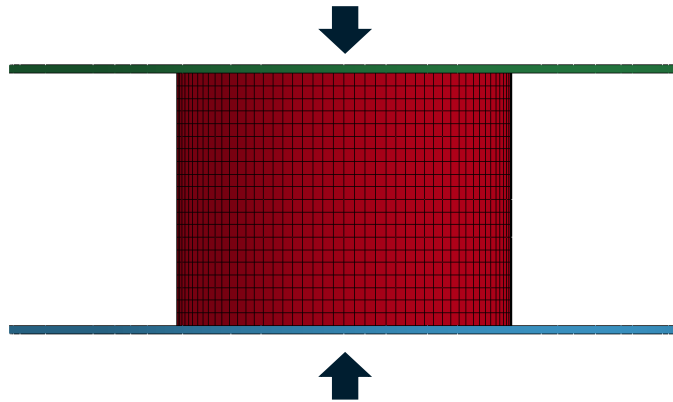
New Model, $\sigma_s = 325$



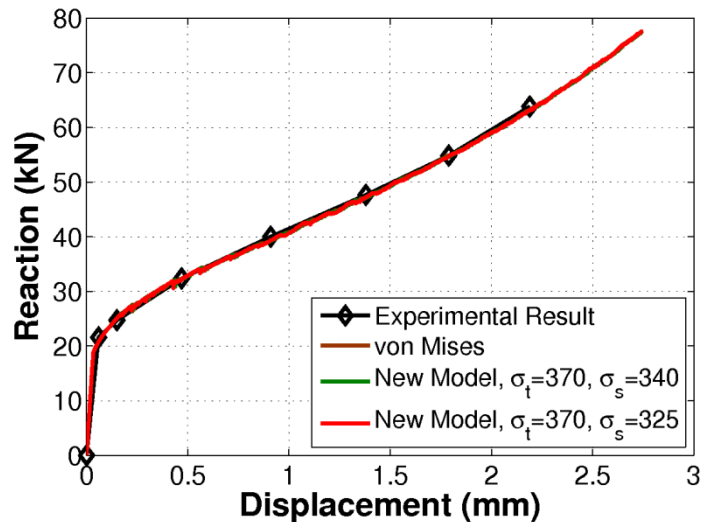


3. Numerical Results

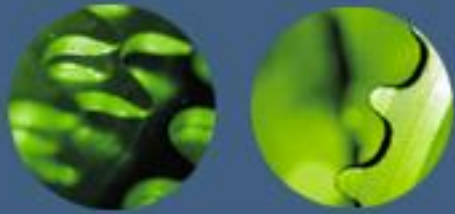
Compressive Stress State: Upset Test



- Number of Elements: 3253
- Friction coefficient: $\mu = 0.05$
- $D = 8\text{mm}$, $h = 6\text{mm}$



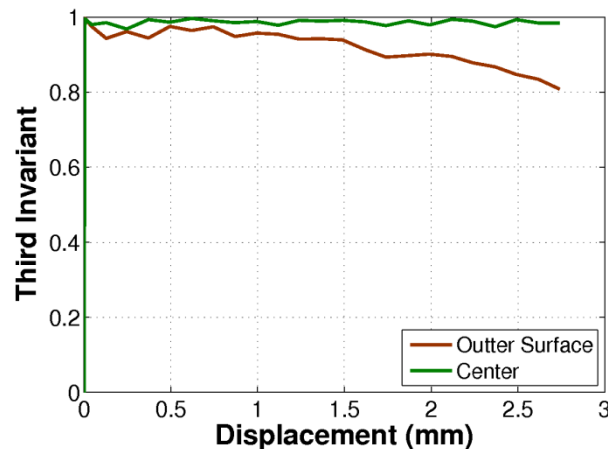
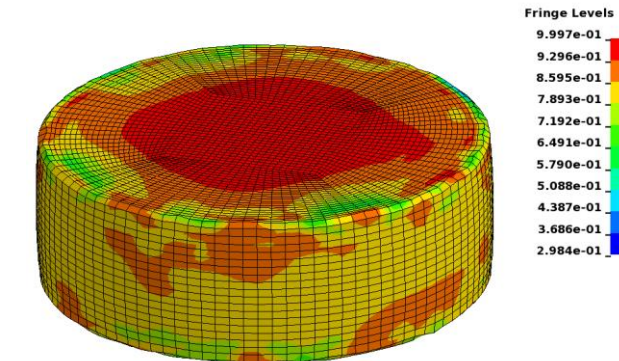
- There is no significant difference between the results provided by the von Mises and the new model
- Results do not change with varying σ_s
- The numerical and experimental results match perfectly



3. Numerical Results

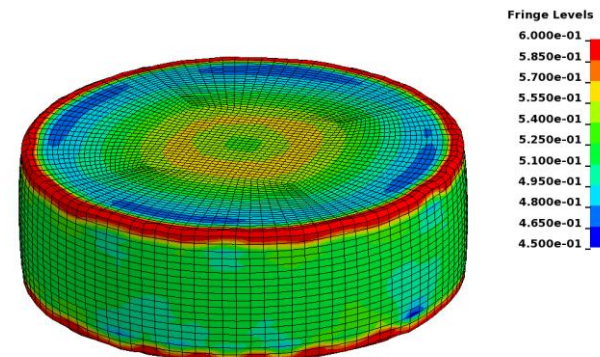
Compressive Stress State: Upset Test

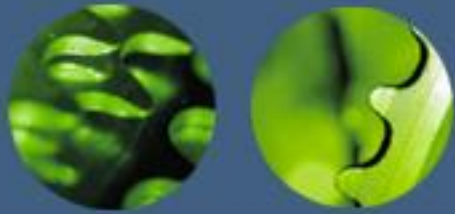
Third Invariant, ξ



- Throughout the deformation process and at the center of the specimen, ξ remains practically constant and equal to 1
- However, on the outer surface, ξ is not constant, verifying a small decrease
- The evolution of ξ on the outer surface of the specimen does not have any impact on the final result for different values of σ_s
- The results are in agreement with the main assumptions initially proposed

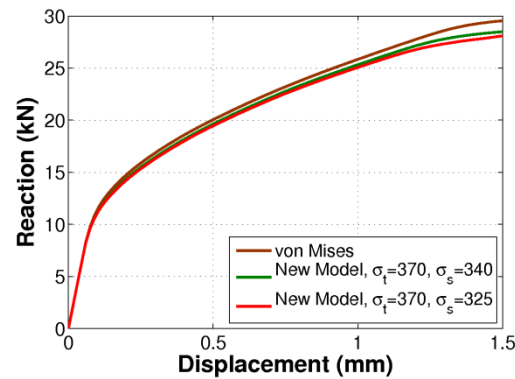
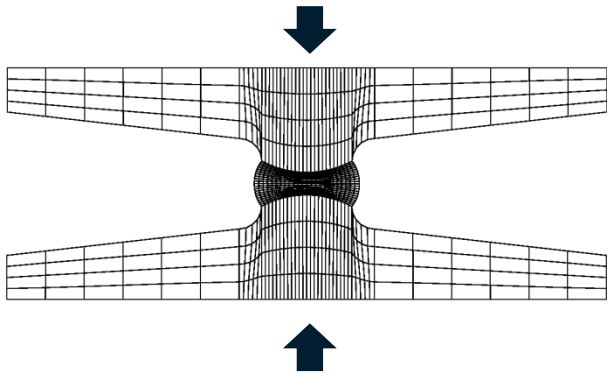
Accumulated Plastic Strain





3. Numerical Results

Compressive Stress State: Butterfly Test

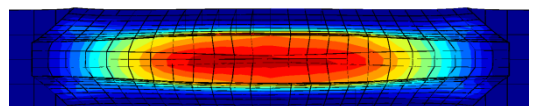
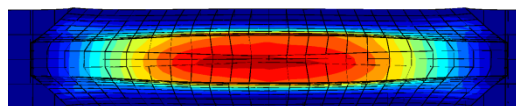
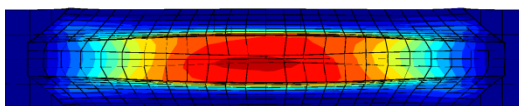
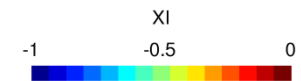
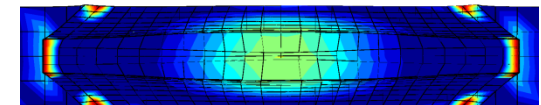
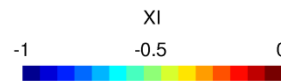
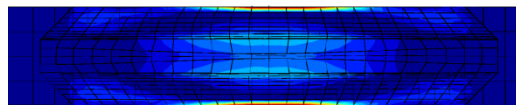
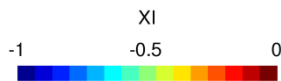
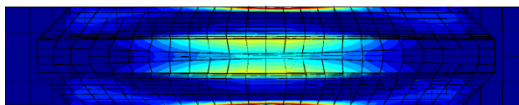


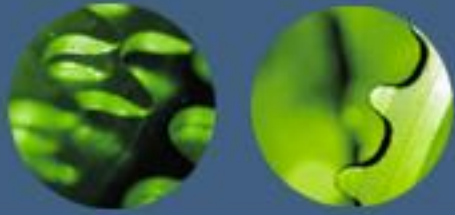
- There is no pronounced difference between the responses provided by von Mises and the new model
- In agreement with the initially desired features

Von Mises

New Model, $\sigma_s = 340$

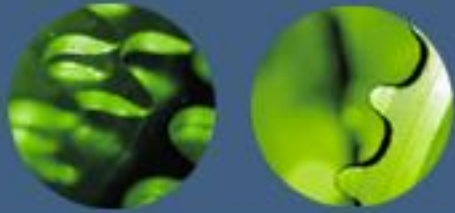
New Model, $\sigma_s = 325$





Final Remarks

- The proposed model is able to capture the effects of the third invariant at different stress states (tension, shear and compression)
- Only one extra material parameter is required by the proposed model when compared to classical von Mises plasticity; the effects of the additional material parameter are also easy to grasp
- From the computational point of view, the proposed model is simple to implement and also to understand; furthermore, it does not present convergence problems
- The initial desired features for the model were achieved



Final Remarks

Future work

- To perform an analytical analysis to find out the critical value of $\sigma_t - \sigma_s$ from which the resulting yield function is non-convex
- To define based on experimental tests how to define σ_s (flat grooved specimen, pure shear test, ...)
- To combine the new model with a damage/failure model (such as GISSMO through *MAT_ADD_EROSION)
- To use the new model to simulate components and structures in practical applications
- To better investigate the yield locus of aluminium alloys based on experimental evidence (is non-convexity perhaps a must in some cases?!)



Thank You!

Visit the city of Porto in Portugal!



fabiojpreis@gmail.com
fjpreis@fe.up.pt