

# DAIMLER

## Modelling of Adhesive Bonding in Crash Simulation

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### Overview

Motivation

FEM modelling of adhesives

- Numerical aspects
- Physical behavior
- Material models

Validation & Verification

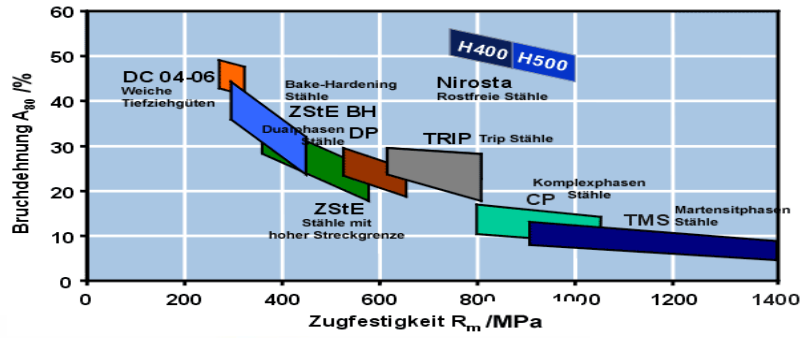
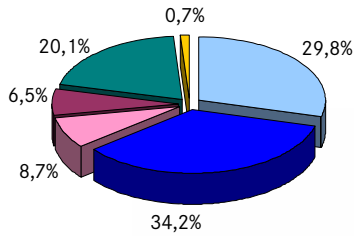
- KS2 specimen tests
- T-component test
- Full car crash test

Summary

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Motivation

S-Class  
Material mix  
(body&white)



- deep drawing steels
- high strength steels
- very high strength steels
- ultra high strength steels
- Aluminium
- Polymers

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Motivation

S-Class  
Joining technology



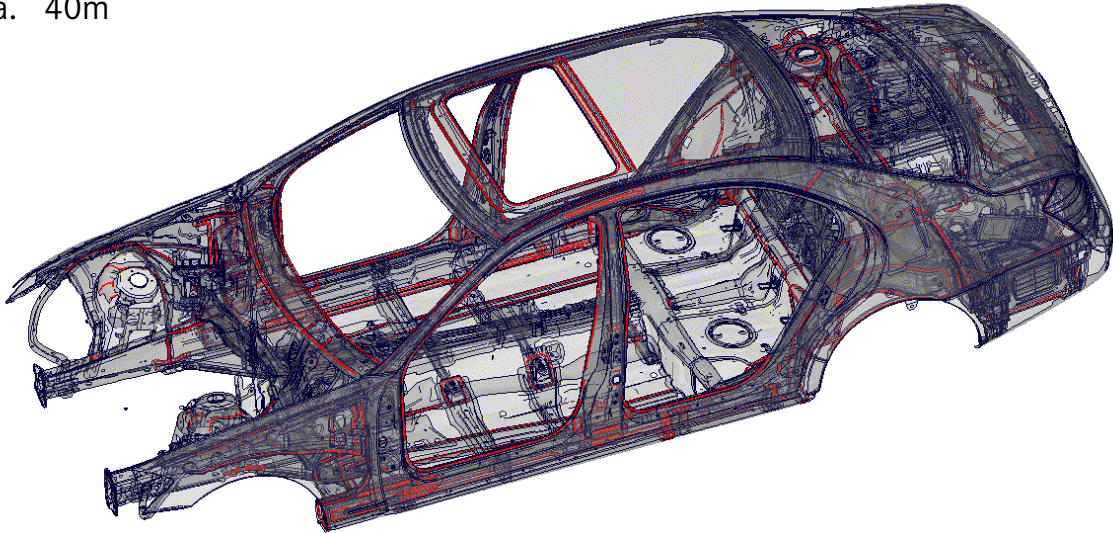
Thermal Joining		BR220	BR221	Mechanical Joining		BR220	BR221
Punktschweißen	[Stk.]	5.486	6.343	Stanznieten	[Stk.]	93	105
Schutzgas-Schweißen	[mm]	1.820	2818	Durchsetzfugen	[Stk.]	153	658
Laser-Schweißen	[mm]	2.600	13.598	Falzen	[mm]	31.600	30.000
Bolzenschweißen	[Stk.]	392	460	Blindnietmuttern	[Stk.]	6	18
Hartlöten	[mm]	340	246	Blindnieten	[Stk.]	10	8
Other Joining		BR220		BR221			
Kleben (Struktur-, Stütz-, und Dichtkleber)		[mm]	72.500	[mm]	191.704		
Nahtabdichtung		[mm]	5.500	[mm]	6.500		
Bauteile		[Stk.]	405	[Stk.]	463		

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## Motivation

S-Class (body&white)  
Structural adhesive

- Structural adhesive  
ca. 150m
- Hood adhesive  
ca. 40m



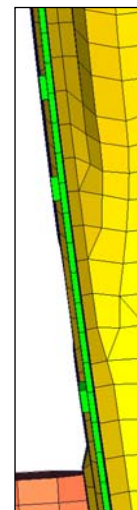
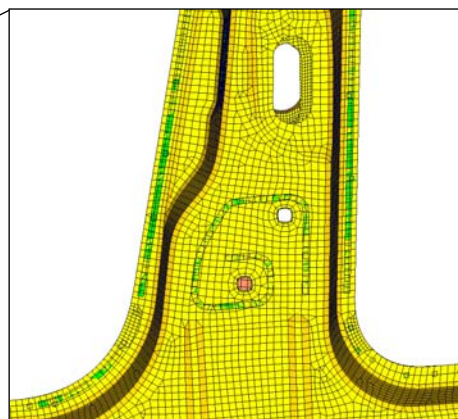
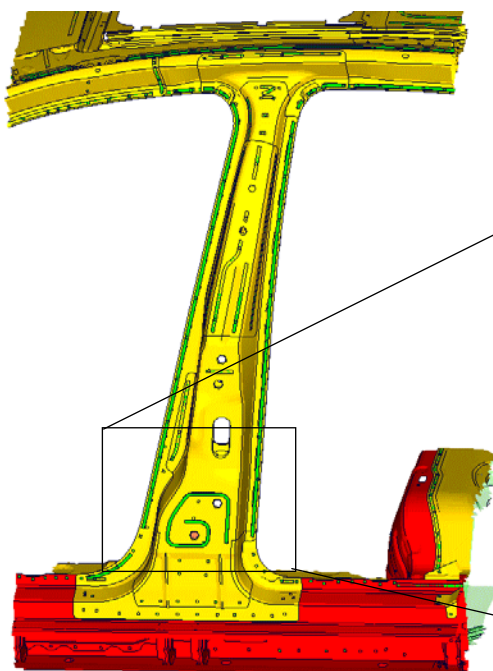
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5

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## FEM modelling: numerical aspects

## Joining in full car crash models



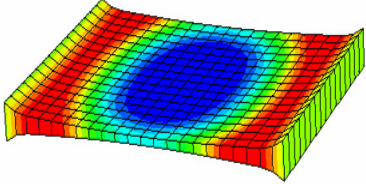
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6

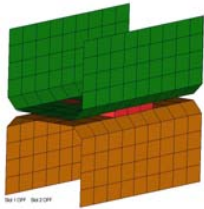
## FEM modelling technique for adhesives

Spatial discretization

- Detailed model

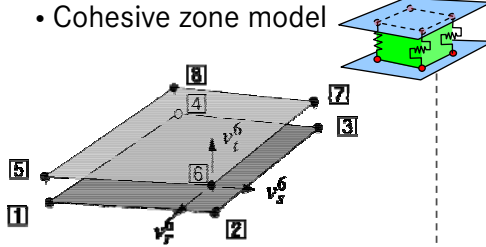


- Substitution model

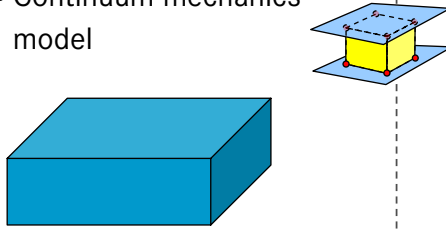


Choice of finite element model

- Cohesive zone model

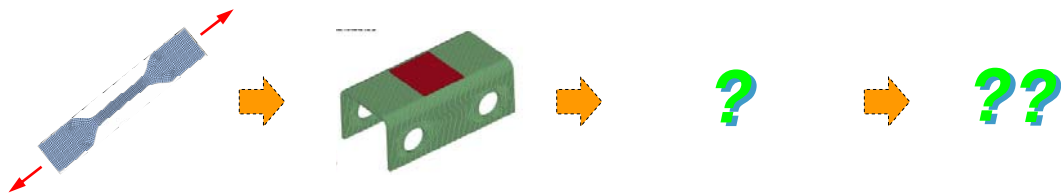


- Continuum mechanics model




## Verification & validation process Consistency

Detailed model with physical material parameters

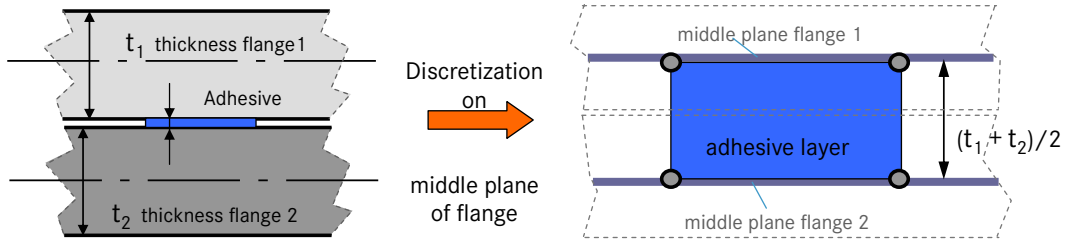


Application of measured physical parameters makes sense only in detailed models!

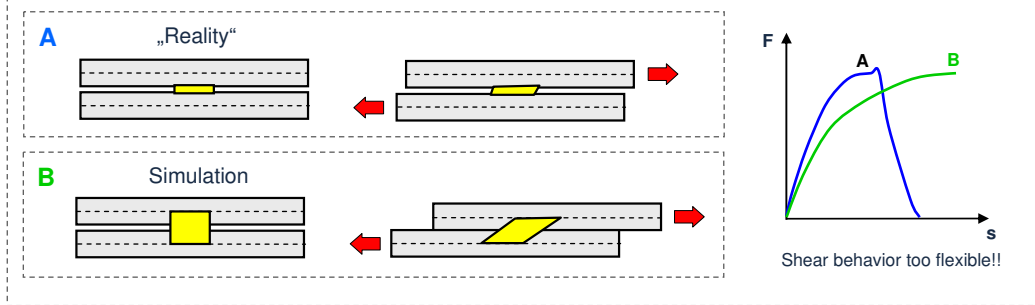
Substitute model with artificial model parameters



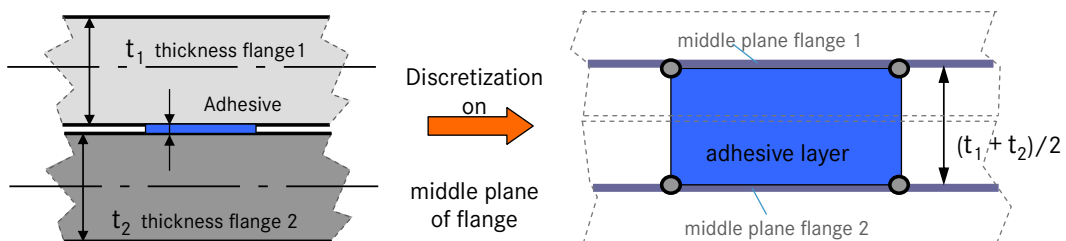
# Adhesive substitution model for crash application



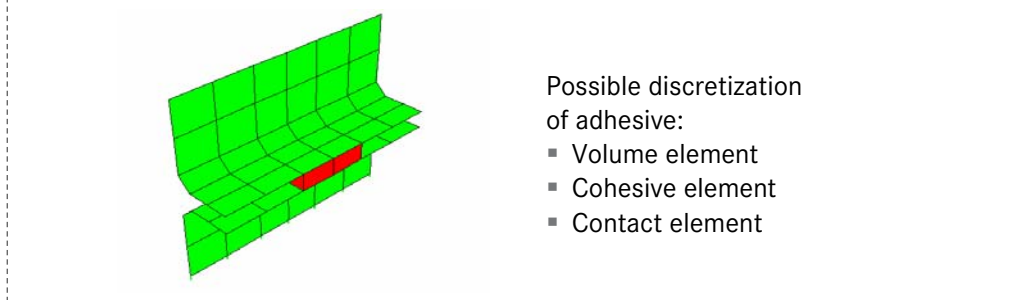
## A discretization issue



# Adhesive substitution model for crash application

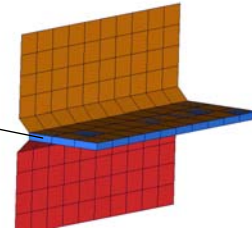


## A discretization issue



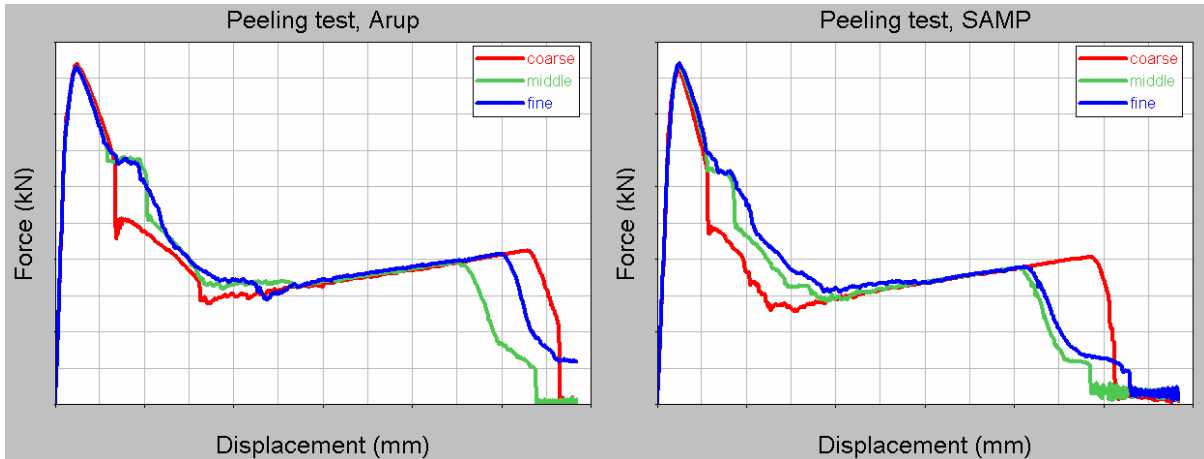
# Influence of mesh size Peeling test

coarse: 3 elements  
middle: 6 elements  
fine: 12 elements  
(only adhesive mesh!)

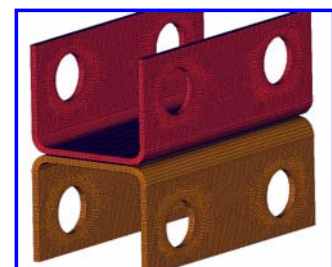
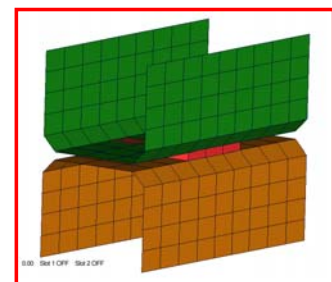
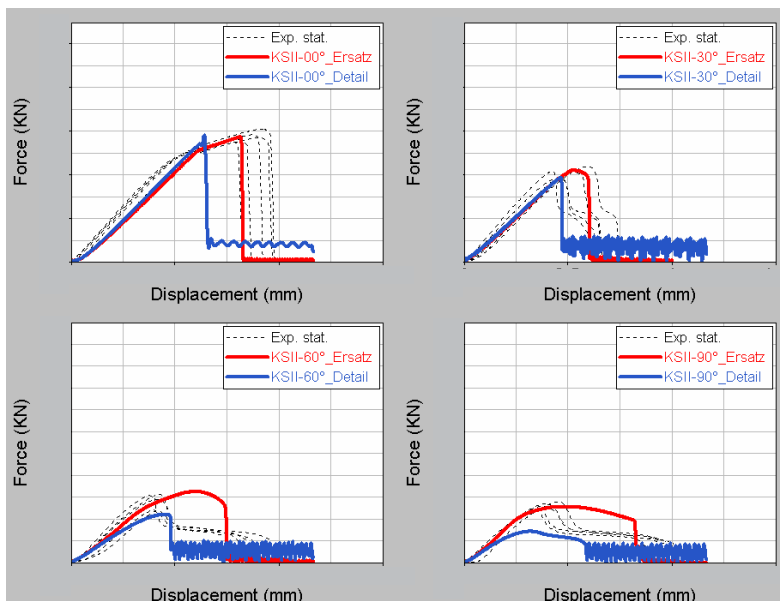


Cohesive model

Volume element



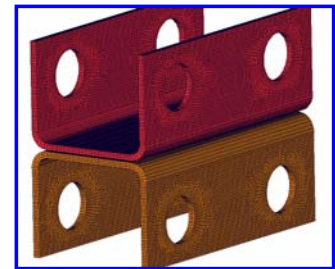
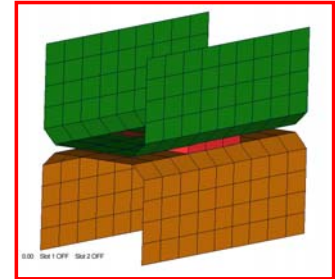
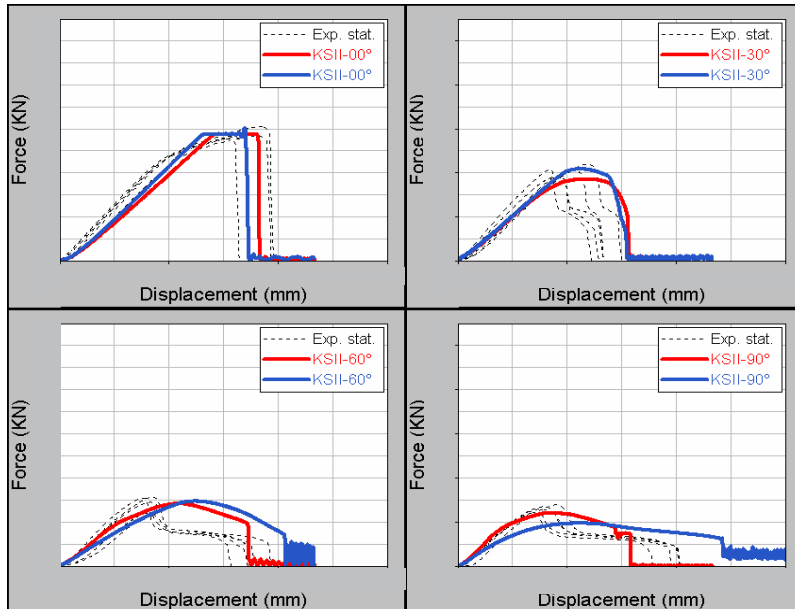
# Comparison detailed vs. substitute model KS2 simulation continuum approach (SAMP model)



### Comparison detailed vs. substitute model

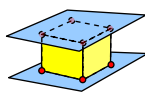
#### KS2 simulation

#### cohesive approach (ARUP model)



### Cohesive vs. continuum approach

#### Continuum element formulation



$$K = \iiint BCB^T \det J d\xi d\eta d\zeta$$

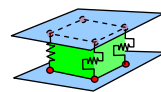
- Integration over volume
- Linear in 3D
- Gaussian integration rule
- Thickness = element length:

$$\boldsymbol{\varepsilon} = \nabla^{sym} \mathbf{u}$$

- Stress vs. strain fully coupled

$$C^{el} = \begin{bmatrix} C_{11} & C_{12} & \dots & C_{16} \\ C_{21} & C_{22} & \dots & C_{26} \\ \dots & \dots & \dots & \dots \\ C_{61} & C_{11} & \dots & C_{66} \end{bmatrix}$$

#### Cohesive (interface) element formulation



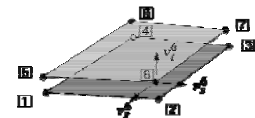
$$K = \iint BCB^T \det J d\xi d\eta$$

- Integration over area
- No in-plane (stress) components
- Lobatto integration rule
- Thickness  $\neq$  element length

$$\boldsymbol{\varepsilon}_z = \frac{\Delta w}{l_c}, \boldsymbol{\varepsilon}_{zx} = \frac{\Delta u}{l_c}, \boldsymbol{\varepsilon}_{zy} = \frac{\Delta v}{l_c}$$

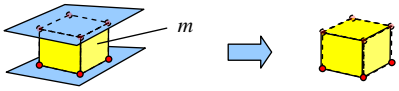
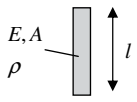
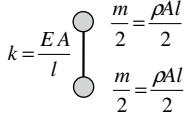
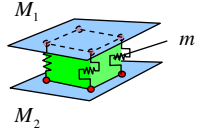
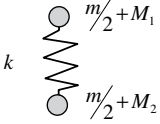
- Traction vs. relative displacement (=test data)

$$C^{el} = \begin{bmatrix} d_r & 0 & 0 \\ 0 & d_s & 0 \\ 0 & 0 & d_t \end{bmatrix}$$

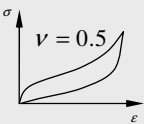
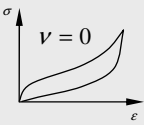
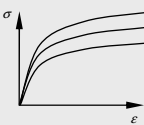



### Critical time step

- Correct time step calculation: consideration of **structural** wave propagation
- Practical use in all FEM-codes: consideration of **elementwise** wave propagation

Continuum element formulation	Cohesive (interface) element formulation
 <p>Consider thickness direction only:</p>  $\Delta t = \frac{l}{c} = l \sqrt{\frac{\rho}{E}}$ <p>Courant-Friedrichs-Levy-criterion (CFL)</p>  $k = \frac{EA}{l}$ $\frac{m}{2} = \frac{\rho Al}{2}$ $\Delta t = \sqrt{\frac{m}{k}} \quad \left( = l \sqrt{\frac{\rho}{E}} \right)$	 <p>very similar to spring elements!</p>  $\Delta t = \sqrt{\frac{m + M_1 + M_2}{k}}$

### Mechanical characterization

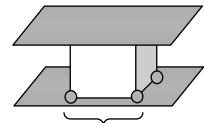
Material behaviour		Choice of material model
<ul style="list-style-type: none"> <li>• PU Windshield =&gt; rubber-like, viscous</li> </ul>		<ul style="list-style-type: none"> <li>• Hyperelastic</li> <li>• Viscous</li> </ul>
<ul style="list-style-type: none"> <li>• Hood =&gt; foam-like, viscous</li> </ul>		<ul style="list-style-type: none"> <li>• Hyperelastic</li> <li>• Viscous</li> </ul>
<ul style="list-style-type: none"> <li>• Structure =&gt; visco-plastic</li> </ul>		<ul style="list-style-type: none"> <li>• Strain rate dependent</li> <li>• Plastic</li> </ul>
		<p>Failure description</p> <ul style="list-style-type: none"> <li>• Johnson-Cook</li> <li>• Energy release rate</li> <li>• Strain or stress based</li> </ul>



## Commonly used material models in LS-DYNA

### Rubber-like adhesives (PU windshield)

- MAT\_SIMPLIFIED\_RUBBER (MAT\_181)  
 $\nu=0.01 \dots 0.499$  (Hill)  
[DuBois/Feng/Kolling 2005]
- MAT\_SIMPLIFIED\_RUBBER\_WITH\_DAMAGE (MAT\_183)  
 $\nu=0.499$  (Ogden)  
[Kolling/DuBois/Benson 2006], work in progress for  $\nu=0.01 \dots 0.499$
- ...



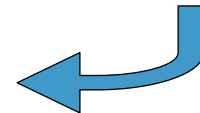
$\nu=0$   
due to constraints

### Foam-like adhesives (Hood)

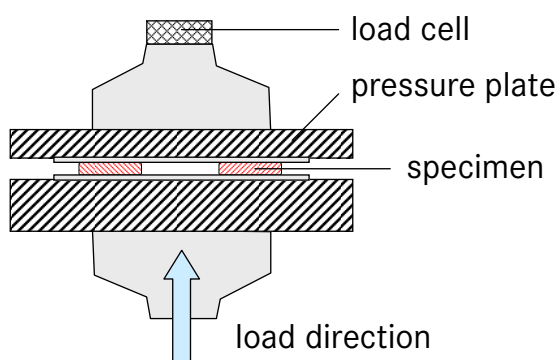
- MAT\_FU\_CHANG\_FOAM (MAT\_83)  $\nu=0$   
modification due to unloading via damage  
[Kolling/Werner/Erhart/DuBois 2007]
- ...

### Visco-plastic adhesives (Structural adhesive)

- MAT\_ARUP\_ADHESIVE
- MAT\_SAMP (MAT\_187)  
[Kolling/Haufe/Feucht/DuBois 2005]
- ...



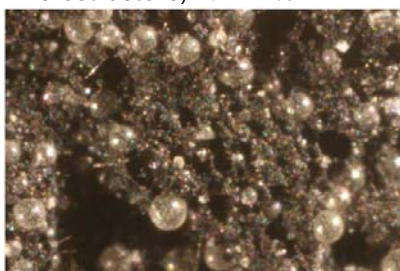
## Foam like adhesive: EFBond



test specimen



Microstructure, 1.7x22.5mm

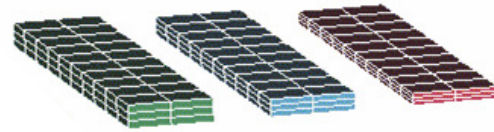


compression

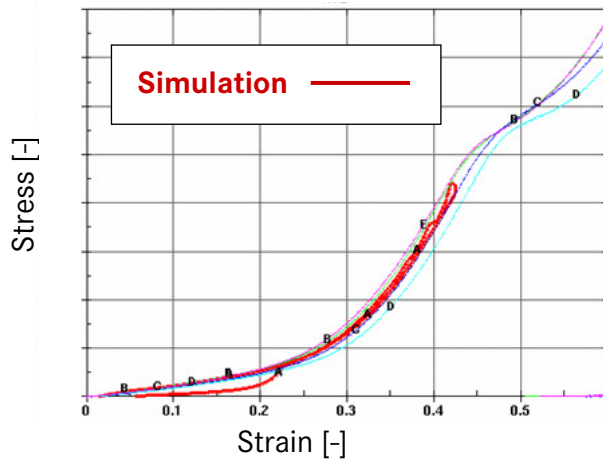


### Foam like adhesive: EFBond

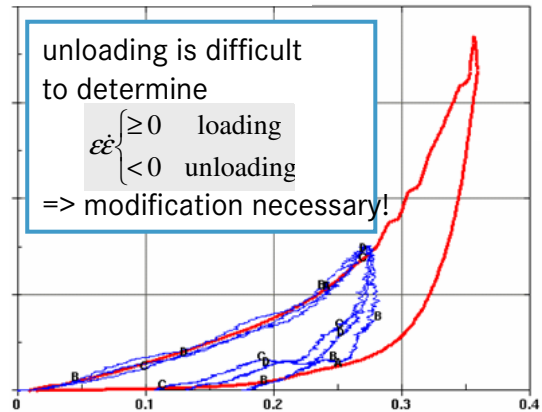
EFBond: MAT\_FU\_CHANG\_FOAM (MAT\_83) in LS-DYNA  
 l=100mm, b=15mm and t=2/3/4 mm  
 volume elements: min l = 0.67mm  
 timestep =  $6.8 \cdot 10^{-6}$  s



quasi static loading

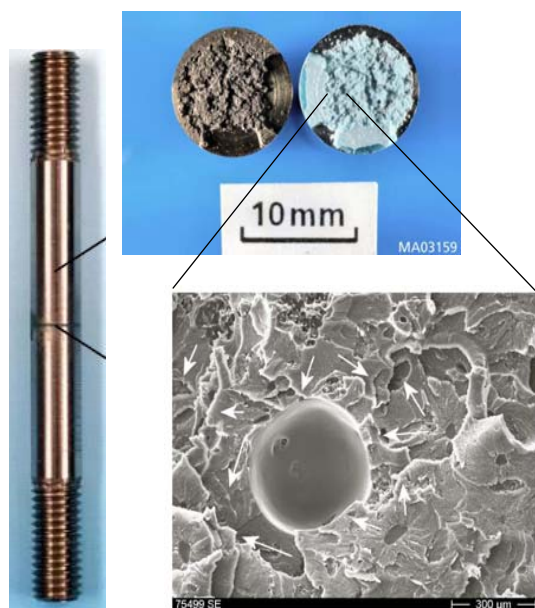


dynamic loading: 800/s

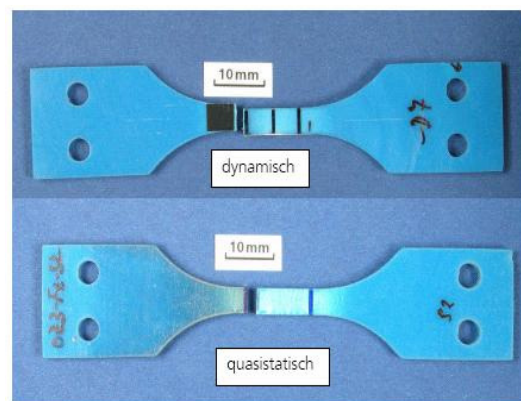


### Structural adhesive: BM 1496 (DOW)

Bonded specimen



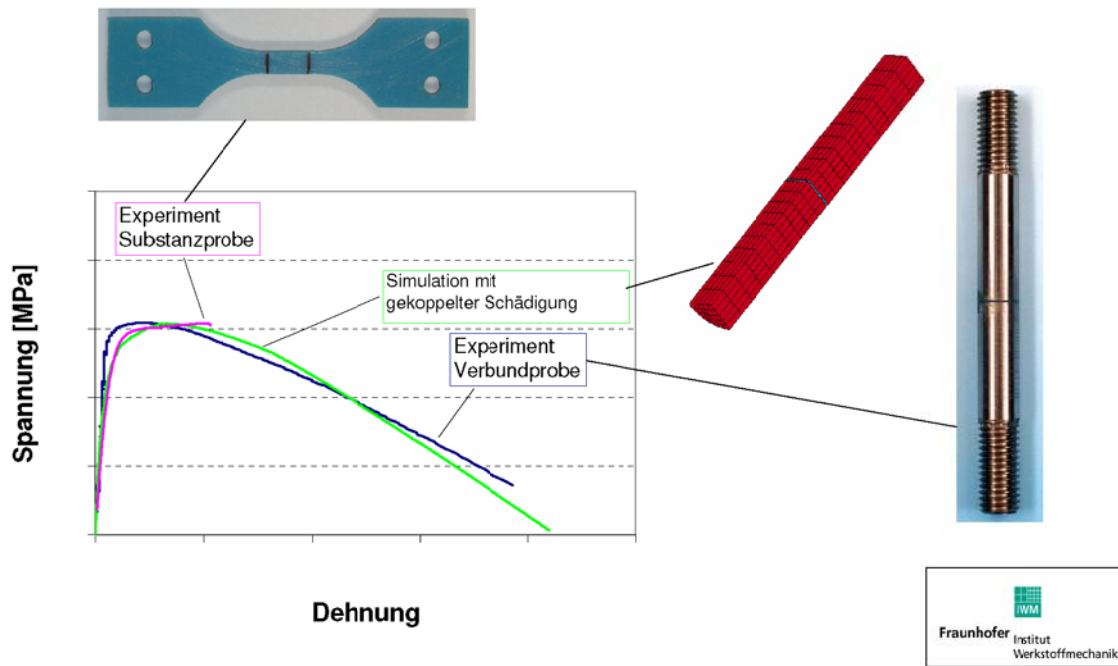
Substanzprobe



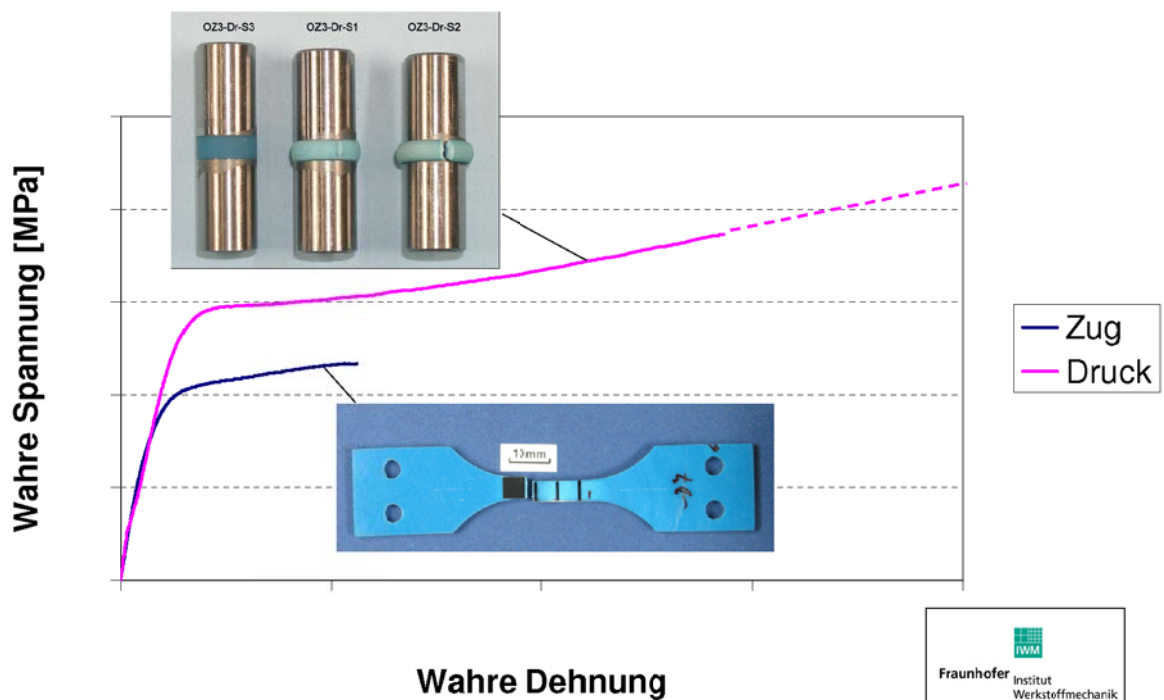
Crazing at static loading



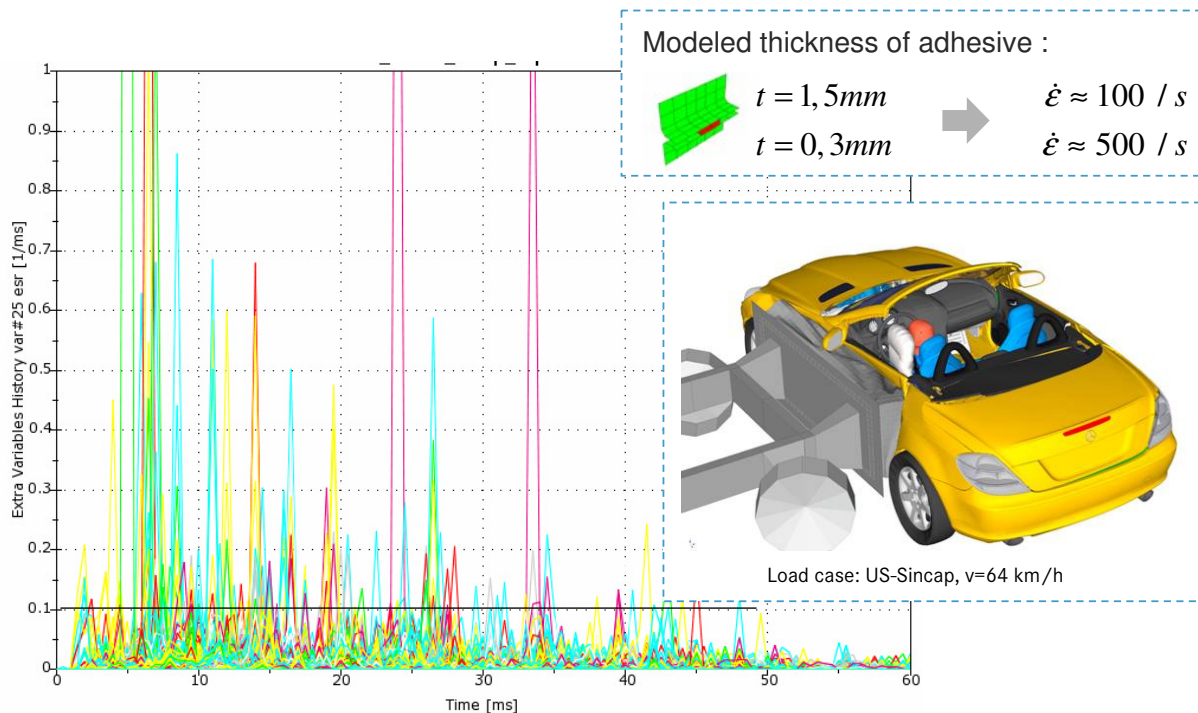
Structural adhesive: BM 1496 (DOW)



Structural adhesive: BM 1496 (DOW)



## Estimate of strain rates from simulation



## Visco-plastic adhesives Investigated models

### Continuum model

- **MAT\_SAMP-1**  
(MAT\_187)
- **MAT\_GURSON**  
(MAT\_120)
- **Fleck Modell**  
(IWM, User material)
- Schlimmer  
(not implemented)
- Spotweld model  
(MAT\_100)

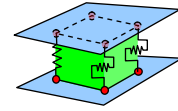
### Cohesive model

- **MAT\_ARUP\_ADHESIVE**  
(MAT\_169)
- **MAT\_COHESIVE\_GENERAL**  
(MAT\_186)
- **MAT\_COHESIVE\_MIXED\_MODE**  
(Matzenmiller, MAT\_138)

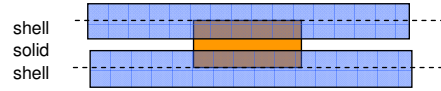
### Contact formulation

- Contact\_One\_Way\_Surface\_  
To\_Surface\_Tiebreak (opt=9)

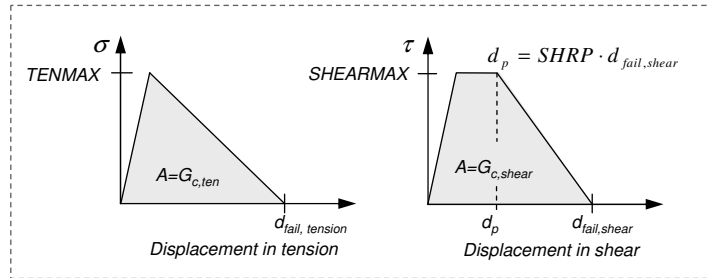
# Cohesive approach MAT\_ARUP\_ADHESIVE



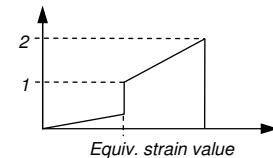
- Modeling with array of solid elements
- Identical solid formulation as in spot weld modeling
- Tied contact in combination with automatic contact
- Failure criterion:



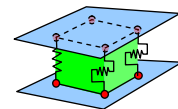
$$\left(\frac{\sigma}{\sigma_{\max}}\right)^{PWRT} + \left(\frac{\tau}{\tau_{\max}}\right)^{PWRS} - 1.0 = 0$$



- Fracture energies for tension and shear as input parameters
- Plastic flow is not volume conserving (not isochoric,  $\nu \neq 0.5$ )
- Special output of equiv. plastic strain if element is fading:  
equiv. pl. strain > 1.0 => Element started fading  
equiv. pl. strain = 2.0 => Element failed



# Cohesive approach MAT\_COHESIVE\_MIXED\_MODE

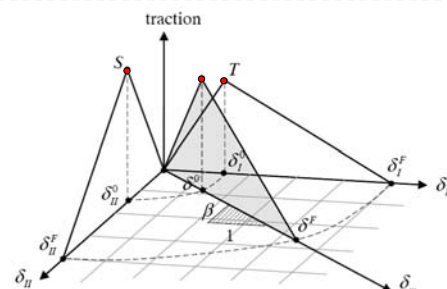


- To be used with cohesive element formulation
- Based on works of Matzenmiller et al.
- Failure occurs when 1 or up to 4 integration points fail (selectable)
- Loading and unloading **do not** follow same path
- Elastic behaviour under compression
- The mixed mode model allows definition of bilinear traction – relative displacement behaviour in mode I (tension) and mode II (shear) loading
- Mode I relative displacement:  $\delta_I^0 = \delta_2^0$ ; mode II relative displacement:  $\delta_{II}^0 = \sqrt{\delta_1^2 + \delta_2^2}$
- Computed mixed mode displacement:  $\delta_m = \sqrt{\delta_I^2 + \delta_{II}^2}$

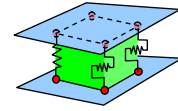
[see Gerlach & Matzenmiller 2006]

• Onset of softening is given  $\delta^0 = \delta_I^0 \delta_{II}^0 \sqrt{\frac{1 + \beta^2}{(\delta_{II}^0)^2 + (\beta \delta_I^0)^2}}$

where  $\beta = \delta_{II} / \delta_I$  "mode mixity"  
 $\delta_I^0 = T / EN$   
 $\delta_{II}^0 = S / ET$  } "peak tractions"



# Cohesive approach MAT\_COHESIVE GENERAL



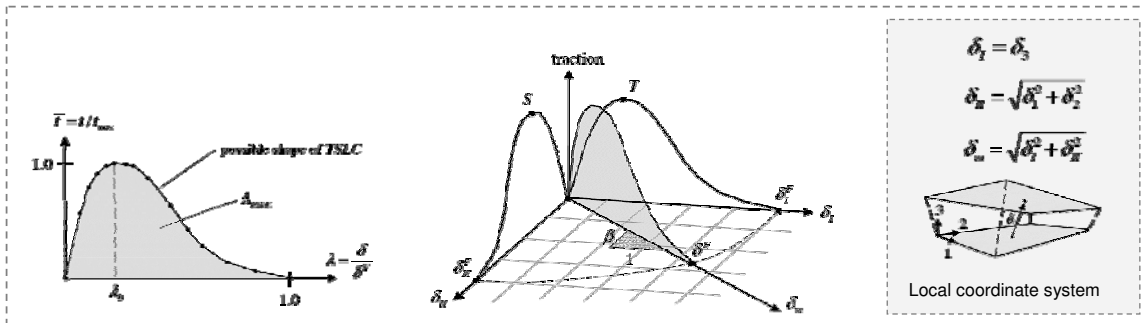
- To be used with cohesive element
- A normalized traction separation curve must be defined. From this, the maximum (failure) separations for mode I and mode II are computed according to:

$$\text{Mode I : } \delta_I^* = \frac{G_I}{A_{TSLC} \cdot T}$$

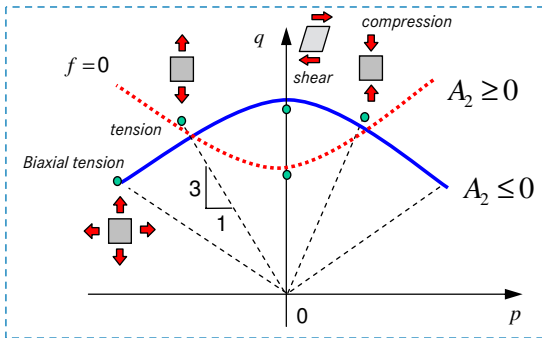
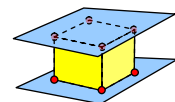
$$\text{Mode II : } \delta_{II}^* = \frac{G_{II}}{A_{TSLC} \cdot S}$$

where A represents the area under the tabulated curve and G the fracture energy release rate of the corresponding mode. T and S are maximum tractions in tension and shear.

- Failure occurs when 1 or up to 4 integration points fail (selectable)
- Loading and unloading **do not** follow same path



# Continuum model MAT\_SAMP (R4)

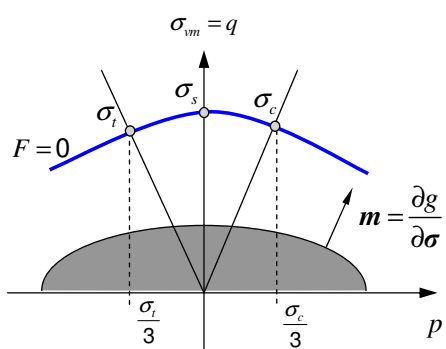


Yield surface (modified Schlimmer):

$$f(p, \sigma_{vm}, \bar{\epsilon}^{pl}) = \sigma_{vm}^2 - A_0 - A_1 p - A_2 p^2 \leq 0$$

Condition for convexity :

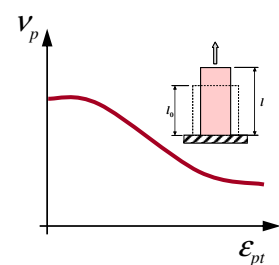
$$A_2 \leq 0 \Leftrightarrow \sigma_s \geq \frac{\sqrt{\sigma_t \sigma_c}}{\sqrt{3}}$$



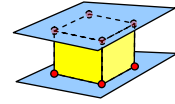
$$g = \sqrt{\sigma_{vm}^2 + \alpha p^2}$$

Flow parameter correlates to plastic Poisson's ratio:

$$\alpha \propto v_p = \frac{9-2\alpha}{18+2\alpha} \leq 0.5$$



Continuum model  
Fleck model (User routine)



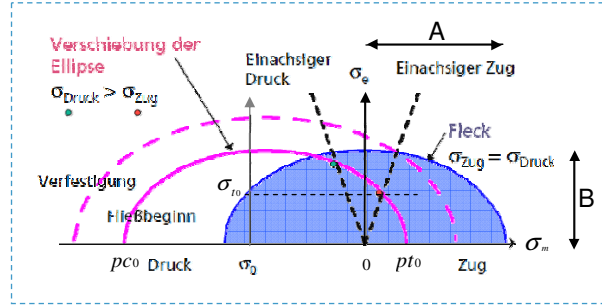
- Yield function

$$\Phi = \hat{\sigma} - \sigma_{t,y}(\bar{\epsilon}_p) \leq 0$$

$$\hat{\sigma}^2 = \sigma_e^2 + \alpha^2 (\sigma_m - \sigma_0)^2$$

$$\alpha = \frac{B}{A} \quad \sigma_0 = \frac{|p_t^0| - |p_c^0|}{2}$$

$$\alpha_0 = \frac{3k_0}{\sqrt{(3k_t^0 + k_0)(3 - k_0)}} \quad \text{and} \quad k_0 = \begin{vmatrix} \sigma_c^0 \\ p_c^0 \end{vmatrix} \quad k_t^0 = \begin{vmatrix} p_t^0 \\ p_c^0 \end{vmatrix}$$



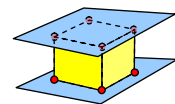
[Source: IWM]

Strain rate dependence via Johnson & Cook:

$$\sigma_t = \sigma_{t0} \cdot [1 + C \cdot \ln(\frac{\dot{\epsilon}}{\dot{\epsilon}_0})]$$



Continuum model  
Fleck model (User routine)

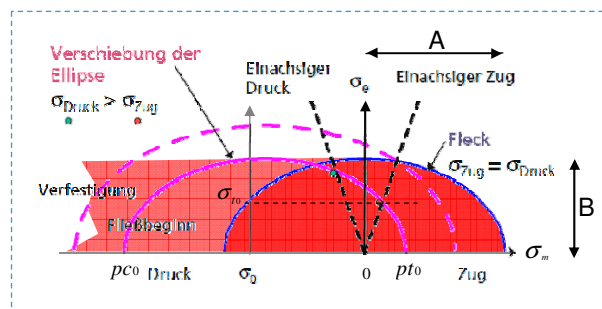


- Plastic Potential

$$G = \sqrt{\sigma_e^2 + \beta^2 \sigma_m^2}$$

$$\beta = \frac{3}{\sqrt{2}} \sqrt{\frac{1 - 2\nu_p}{1 + \nu_p}}$$

$$\Leftrightarrow \nu_p = -\frac{\Delta \epsilon_{11}^p}{\Delta \epsilon_{33}^p} = \frac{\frac{1}{2} - (\frac{\beta}{3})^2}{1 + (\frac{\beta}{3})^2}$$



[Source: IWM]

i.e.  $\beta$  determines plastic Poisson's ratio (dilatancy!).

If  $\beta < 0$ , potential switches to vonMises type.

- Failure criterion

Johnson & Cook (with strain rate dependency):

$$\bar{\sigma}_f^p = \left[ d_1 + d_2 \exp\left(d_3 \frac{p}{q}\right) \right] \left[ 1 + d_4 \ln\left(\frac{\dot{\epsilon}_p}{\dot{\epsilon}_0}\right) \right]$$

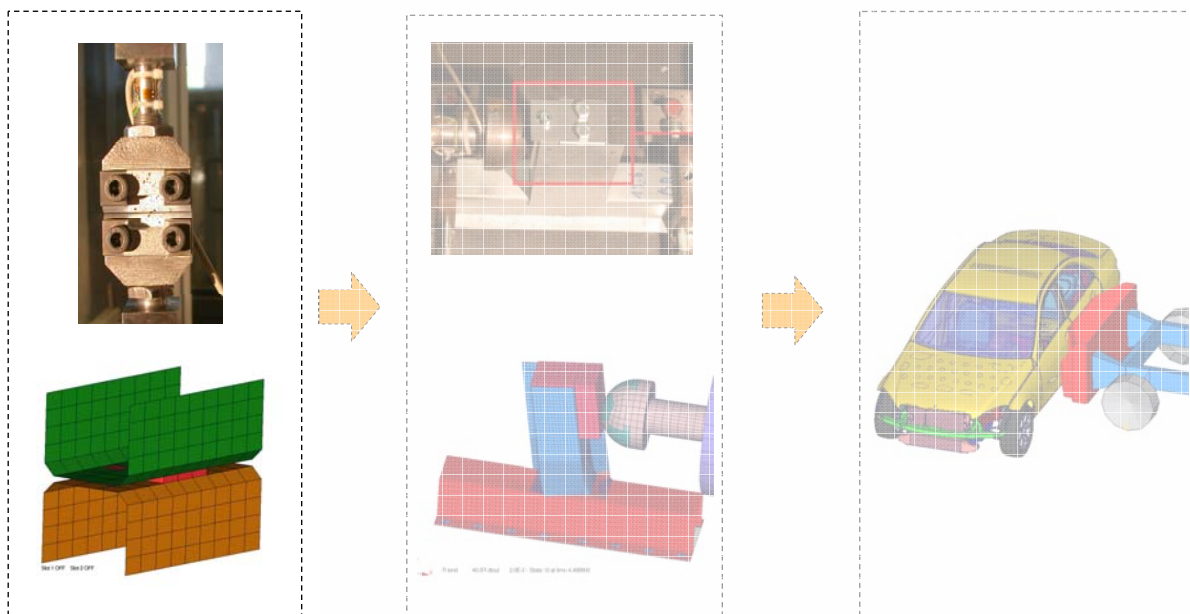
## Continuum theory based models

### Comparison

	Schlimmer	IWM-Fleck	SAMP-1
Yield function	$f = \sigma_{vm}^2 + 9a_2(p - \sigma_0)^2 - Y^2$	$f = \sqrt{\sigma_{vm}^2 + \alpha^2(p - \sigma_0)^2} - Y$	$f = \sigma_{vm}^2 - A_0 - A_1p - A_2p^2$
Plastic potential	$g = \sigma_{vm}^2 + \beta^2 p^2$	$g = \sqrt{\sigma_{vm}^2 + (\beta)^2 p^2}$	$g = \sigma_{vm}^2 + \alpha p^2$
Yield stress			
- Reference	Shear / Torsion	Shear/Compression/ Tension	Shear/Compression/ Tension/Biaxial
- Defined by	Parameter	Parameter	Tabulated
- Strain rate	✘	✓	✓
Failure model	✘	Johnson-Cook	tabulated regularized

## Verification & validation process

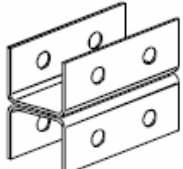
### KS2 test verification

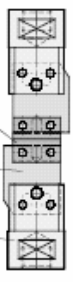




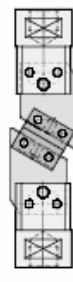
Verification experiments  
KS2 tests

**KS2 Specimen**  
90°, 60°, 30°, 0°

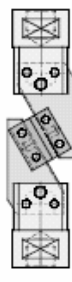




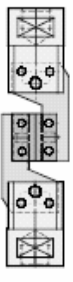
Aufnahme 90°



Aufnahme 60°



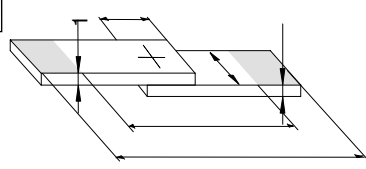
Aufnahme 30°



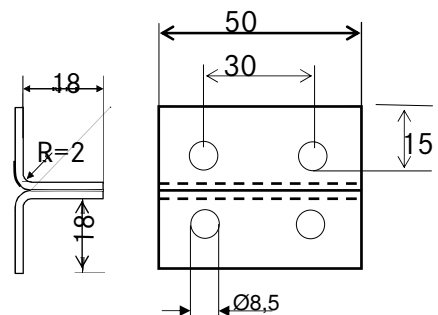
Aufnahme 0°

Prüfkörper  
Prüfkörper-  
aufnahme  
Adapter

**Shear**



**Peel-test**



6th LSDYNA-Forum, Frankenthal / Dr. Feucht (EP-SPB) / 2007-10-11

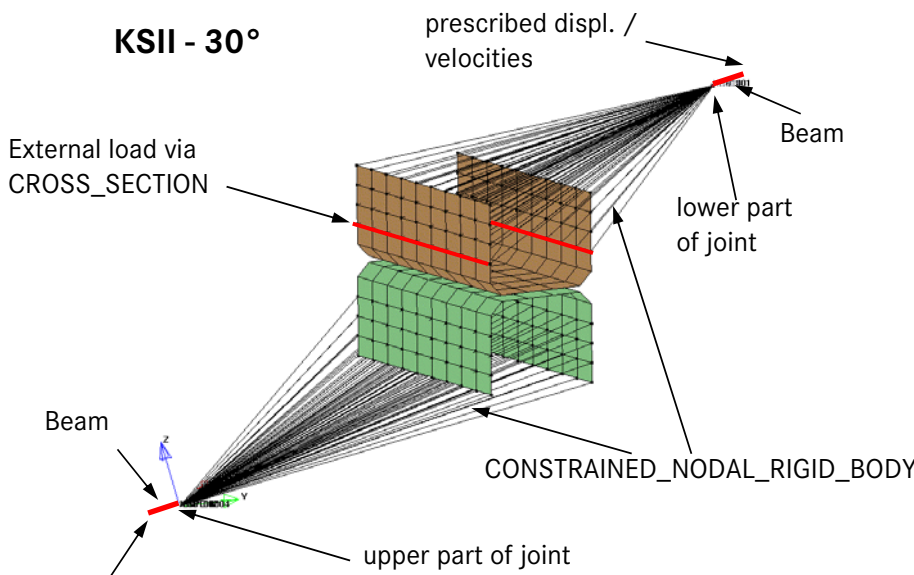
33

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Validation & Verification: KS2 tests

KSII test configuration  
Crash consistent simulation model

**KSII - 30°**



prescribed displ. / velocities

External load via CROSS\_SECTION

Beam

lower part of joint


CONstrained\_NODAL\_RIGID\_BODY

Beam

upper part of joint

all translative degrees of freedom fixed

**KSII - 0°**



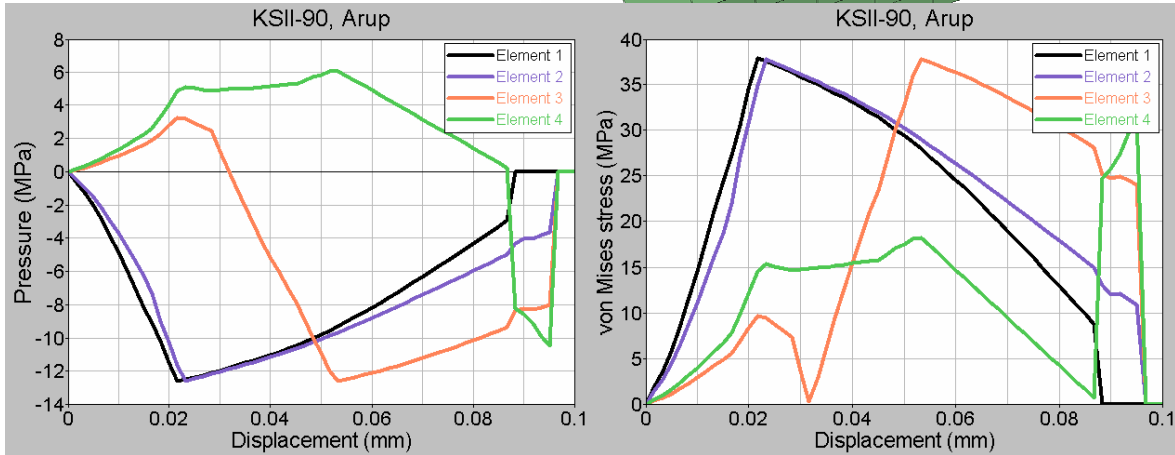
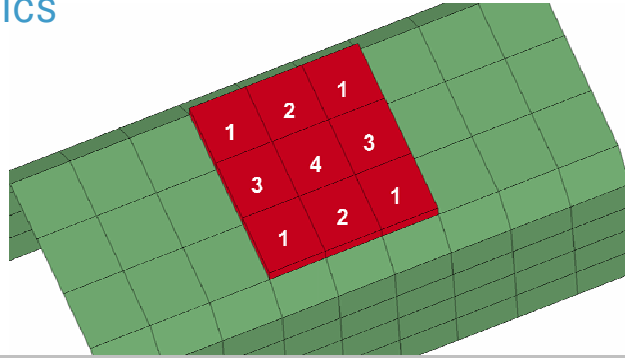
LWF  
Uni Paderborn

6th LSDYNA-Forum, Frankenthal / Dr. Feucht (EP-SPB) / 2007-10-11

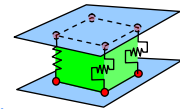
34

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Cross tension characteristics  
KS2 - 90°

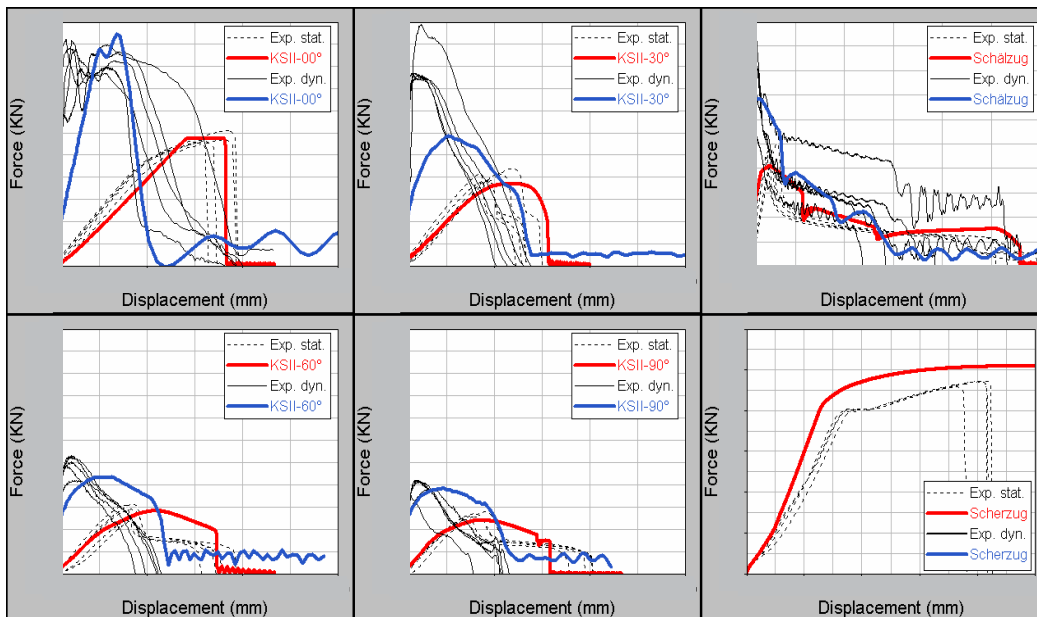


KS2 simulation results  
ARUP model

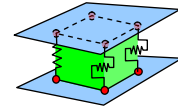


1 strain rate dependent material card!

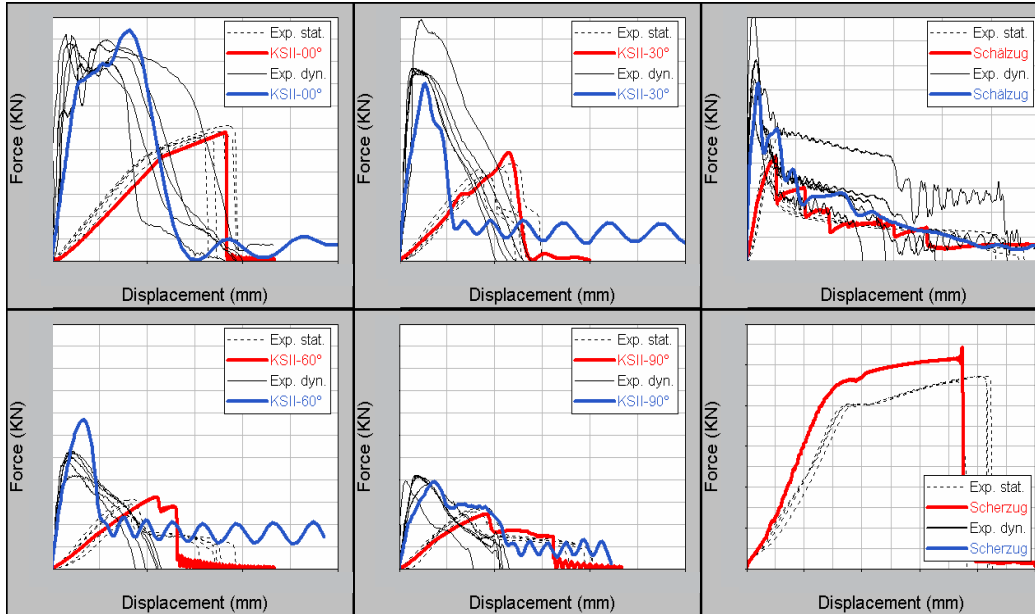
dynamic loading (2.5 m/s)  
quasi-static loading



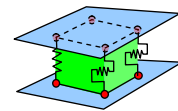
# KS2 simulation results Cohesive general model



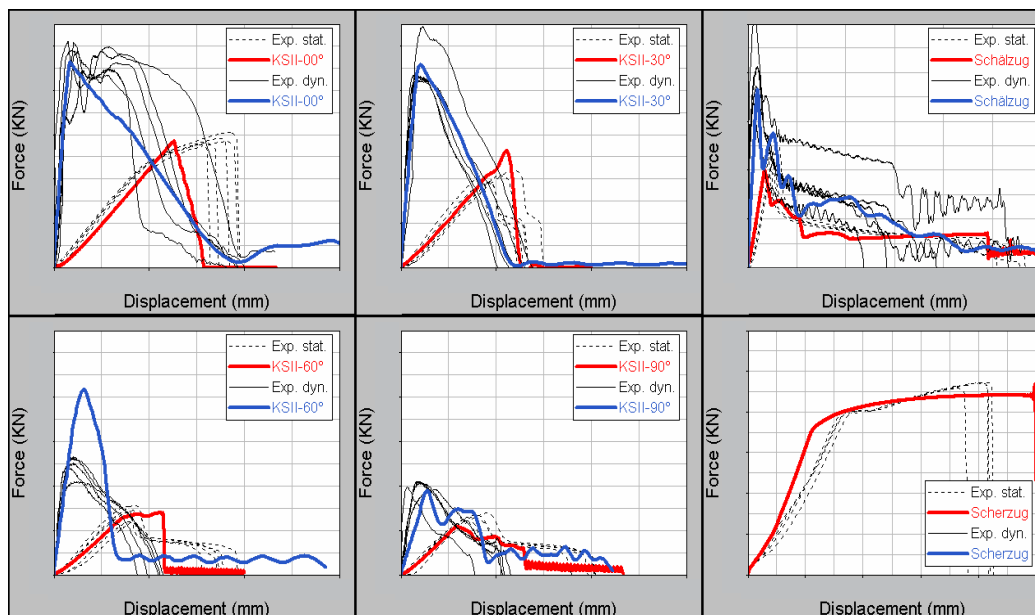
2 Materialcards, because model is not strain rate dependent!



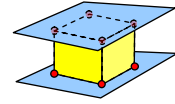
# KS2 simulation results Cohesive mixed mode model



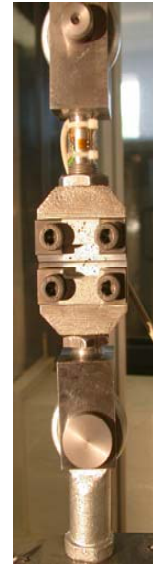
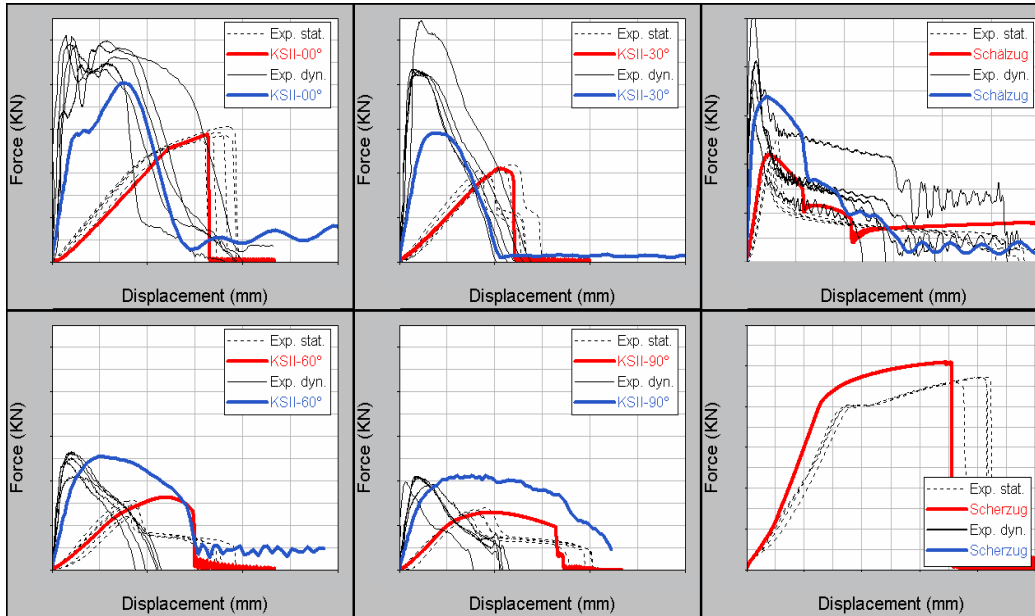
2 Materialcards, because model is not strain rate dependent!



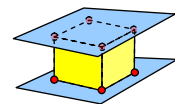
# KS2 simulation results SAMP model (R4)



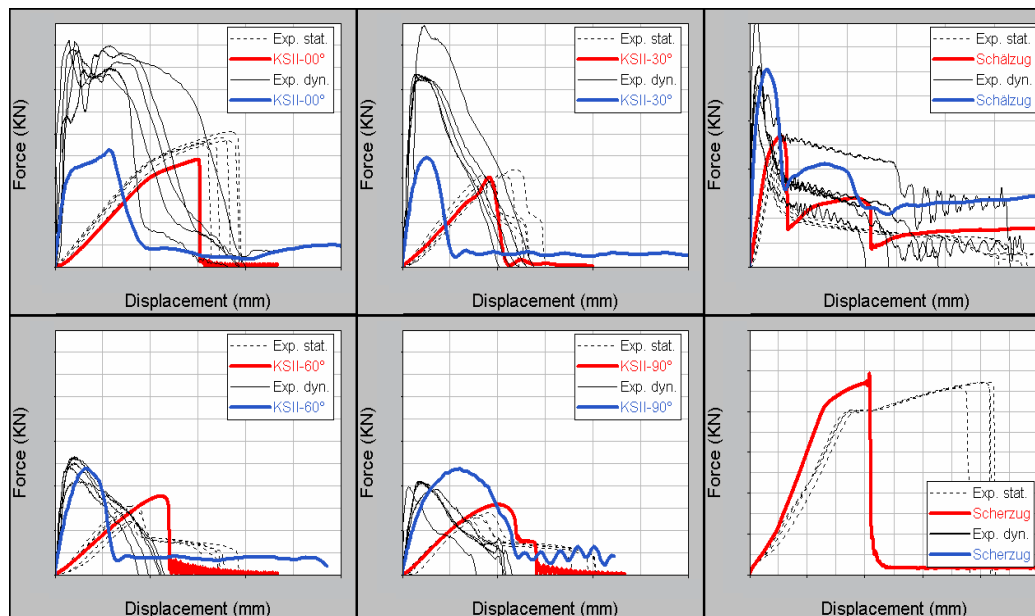
1 strain rate dependent material card!



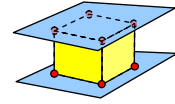
# KS2 simulation results Fleck model



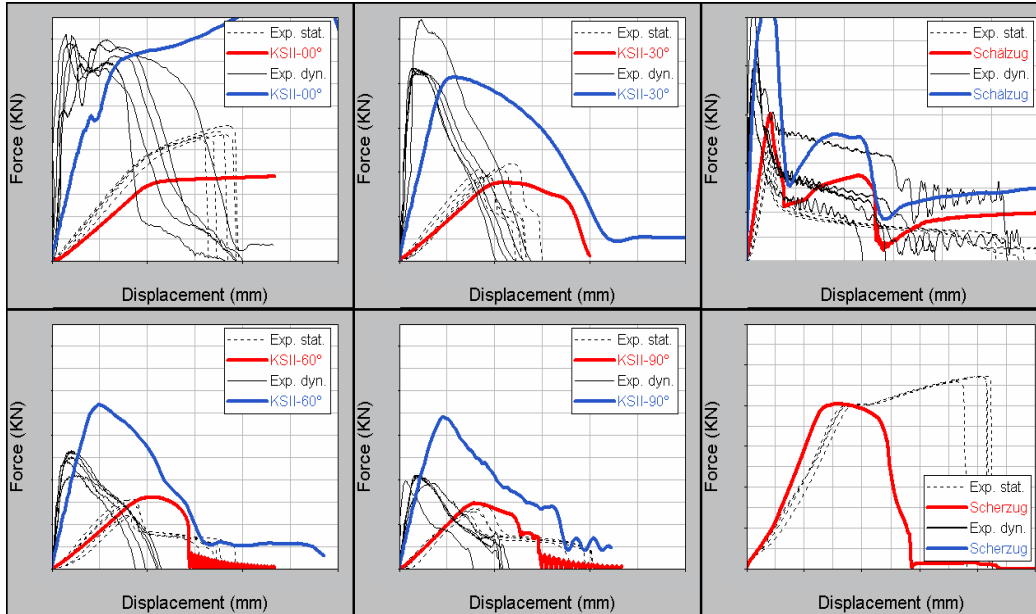
1 strain rate dependent material card!



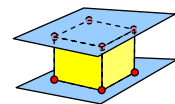
# KS2 simulation results Gurson model



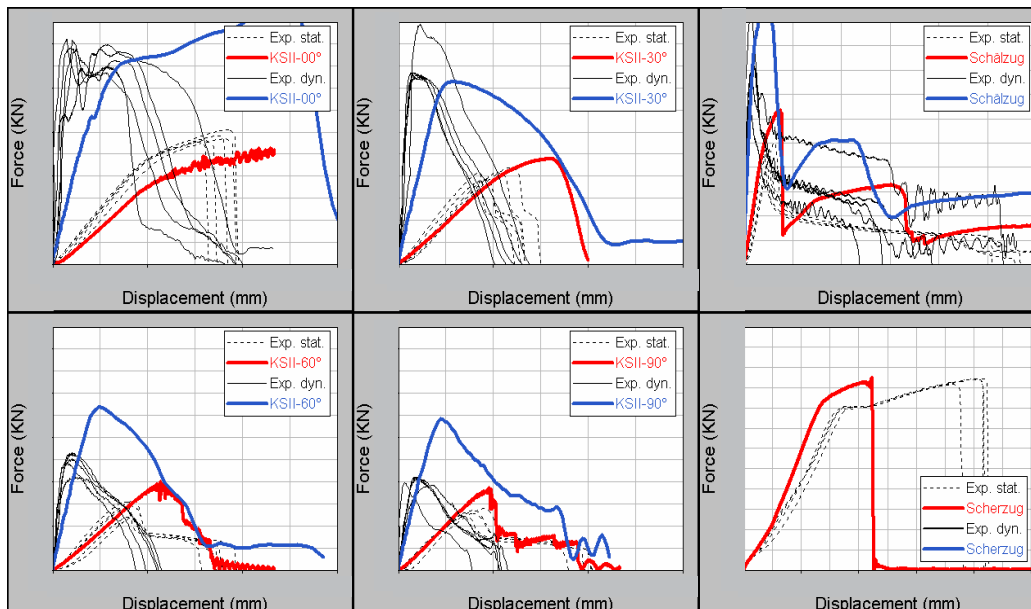
1 strain rate dependent material card!



# KS2 simulation results Gurson-Johnson-Cook model

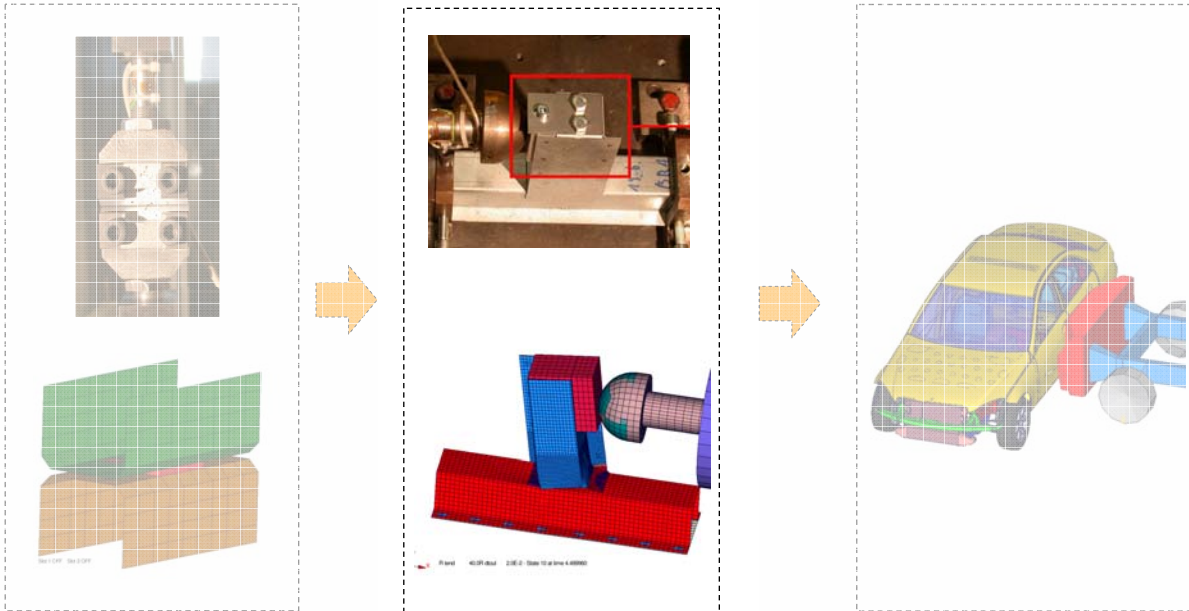


1 strain rate dependent material card!



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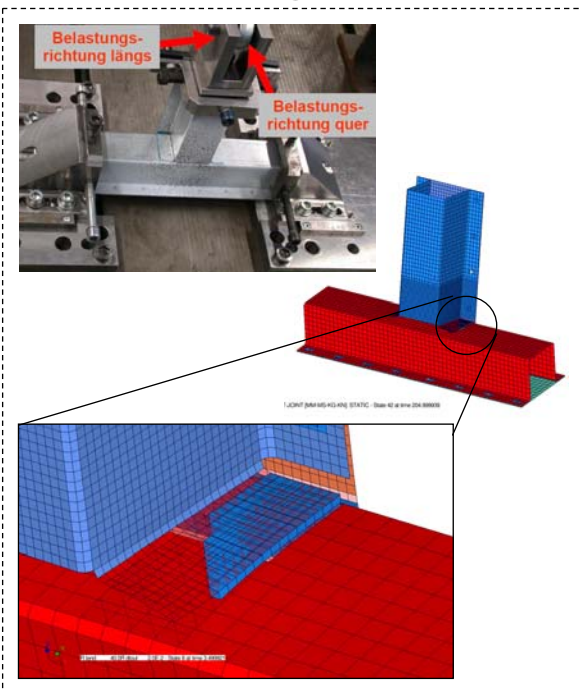
Verification & validation process  
T-component validation



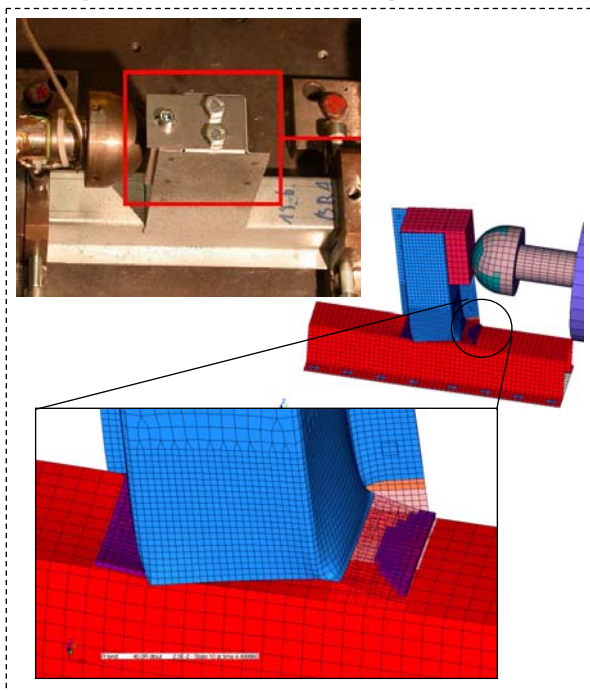
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Validation & Verification: T-component

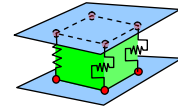
T-component tests  
Lateral loading



longitudinal loading

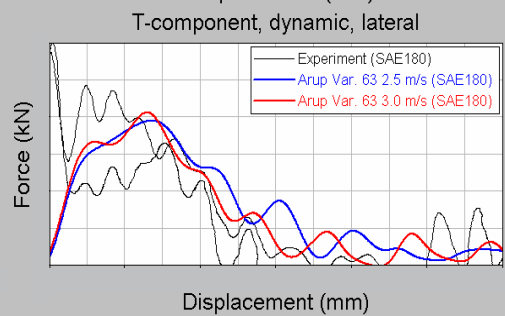
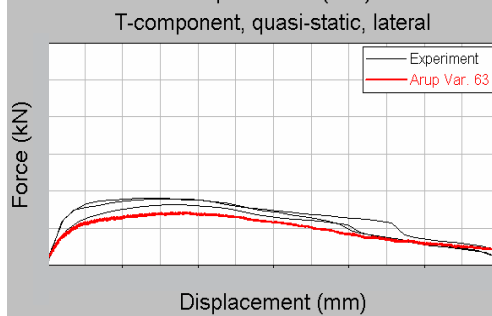
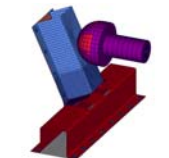
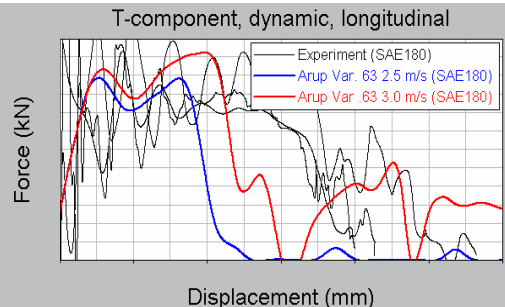
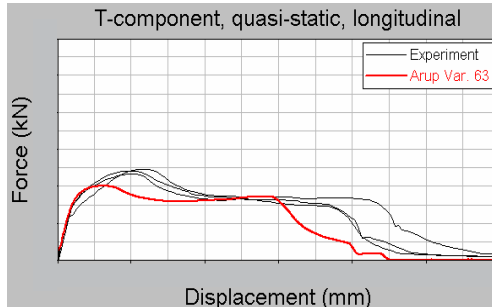
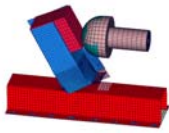


# T-component simulation results ARUP model

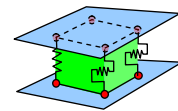


quasistatic

dynamic loading

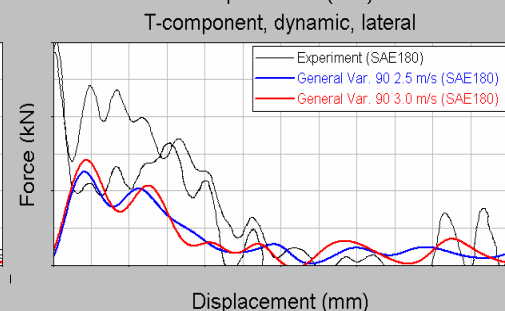
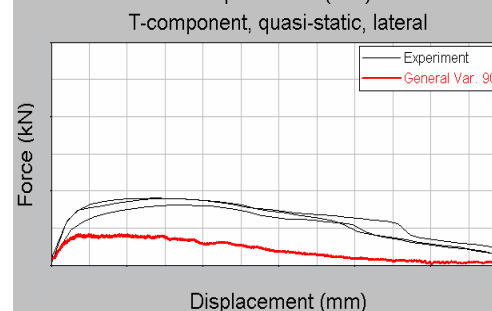
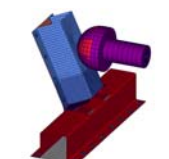
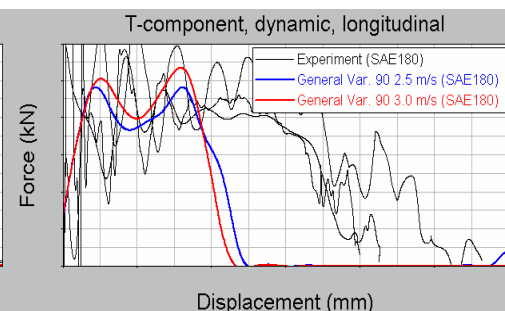
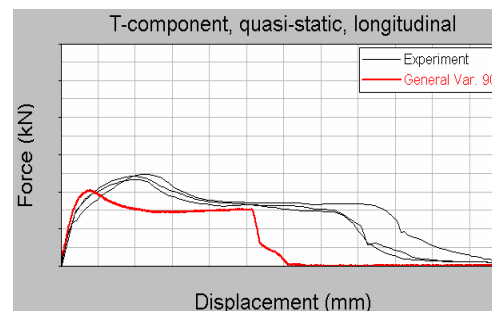
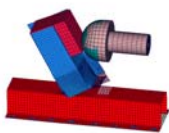


# T-component simulation results Cohesive general model



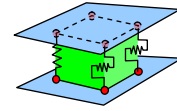
quasistatic

dynamic loading



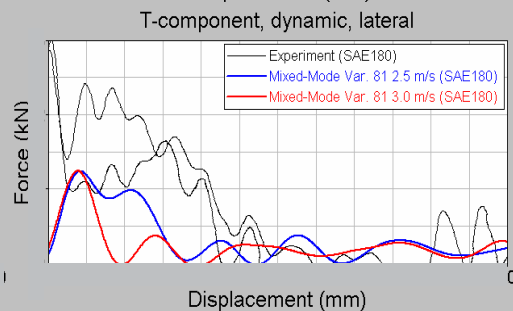
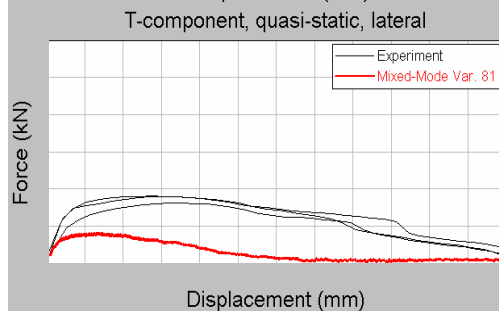
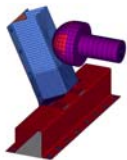
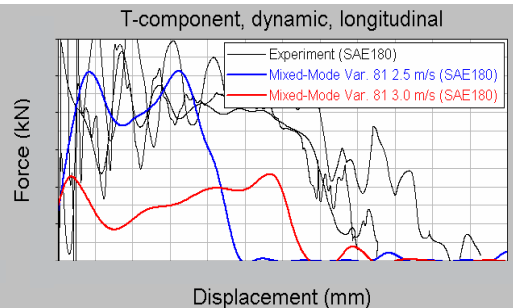
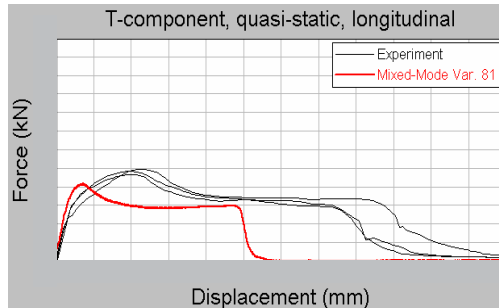
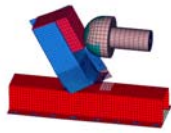
# T-component simulation results

## Cohesive mixed mode model



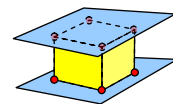
quasistatic

dynamic loading



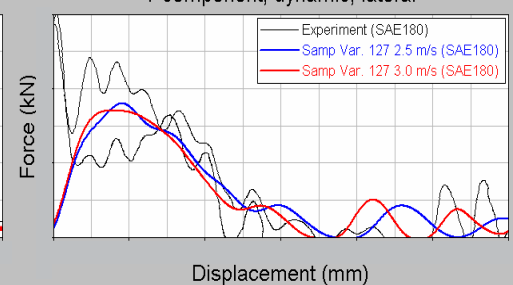
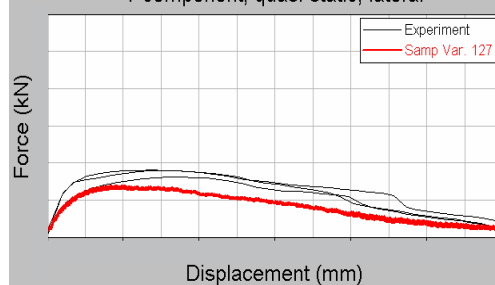
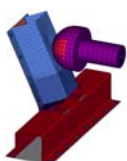
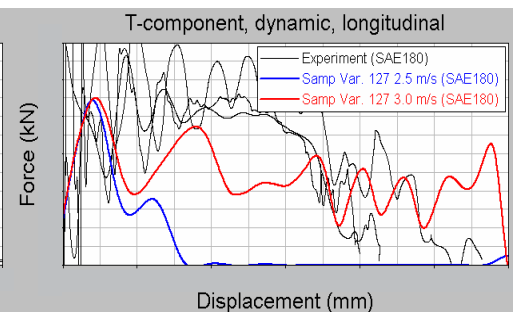
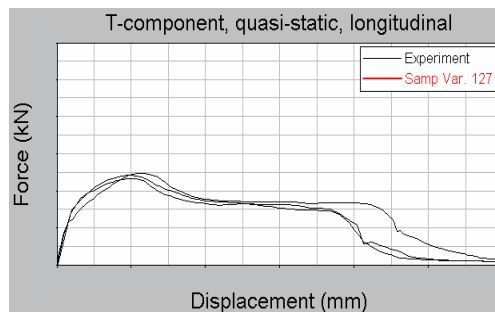
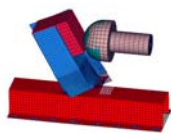
# T-component simulation results

## SAMP model



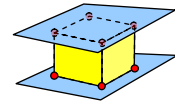
quasistatic

dynamic loading



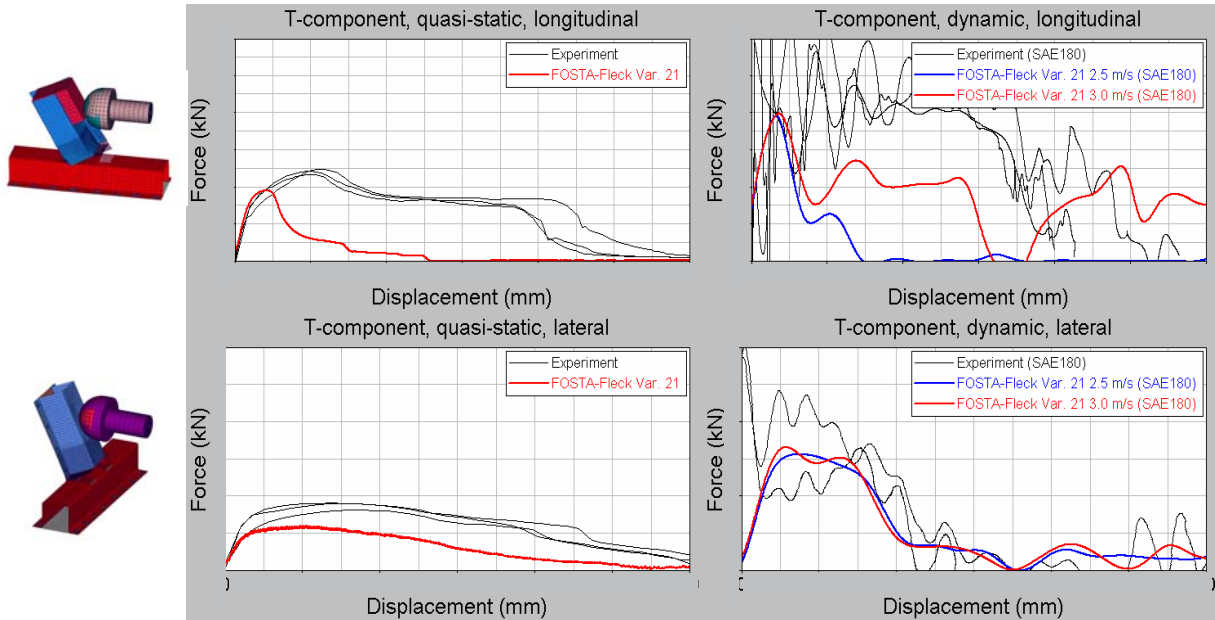


# T-component simulation results Fleck model

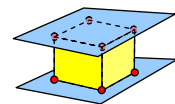


quasistatic

dynamic loading

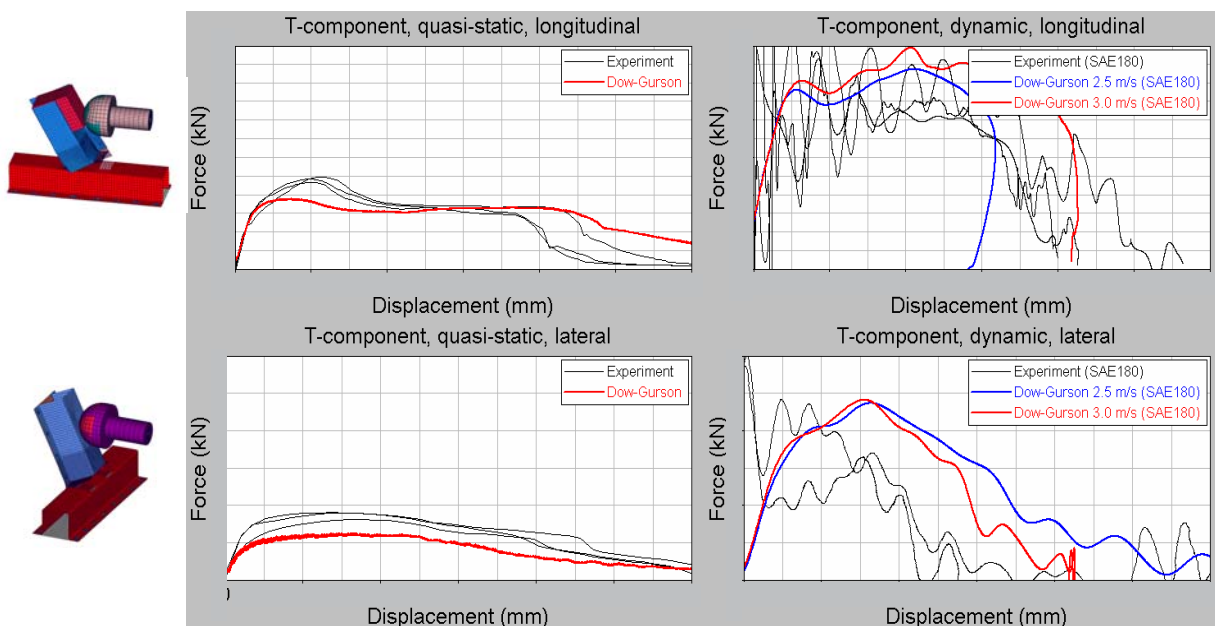


# T-component simulation results Gurson model

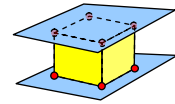


quasistatic

dynamic loading

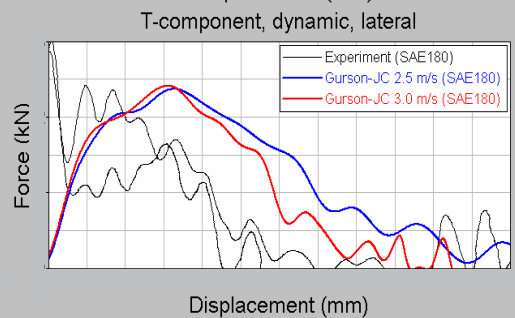
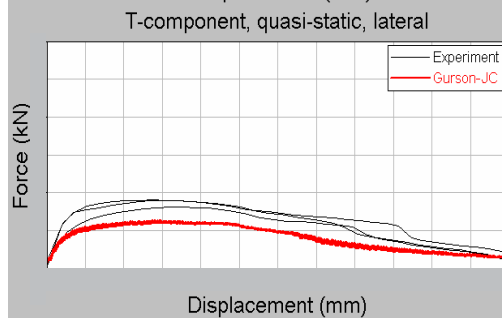
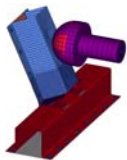
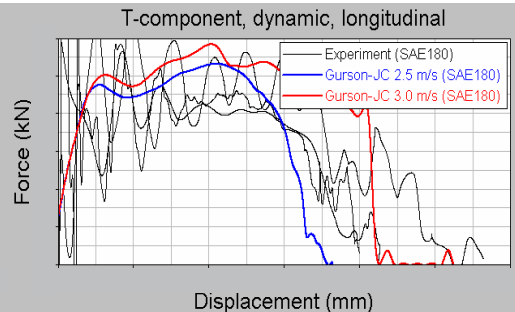
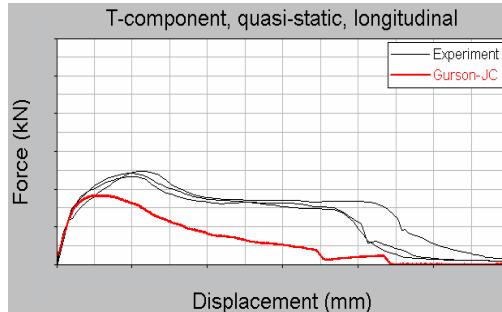
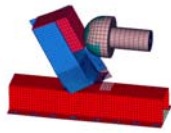


# T-component simulation results Gurson-Johnson-Cook model

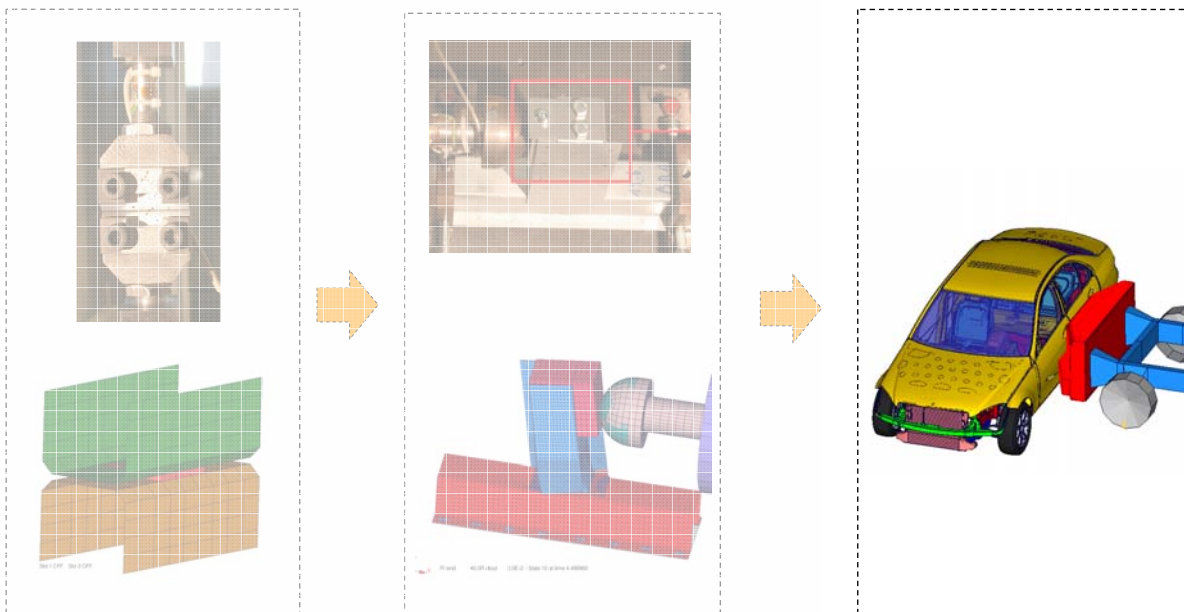


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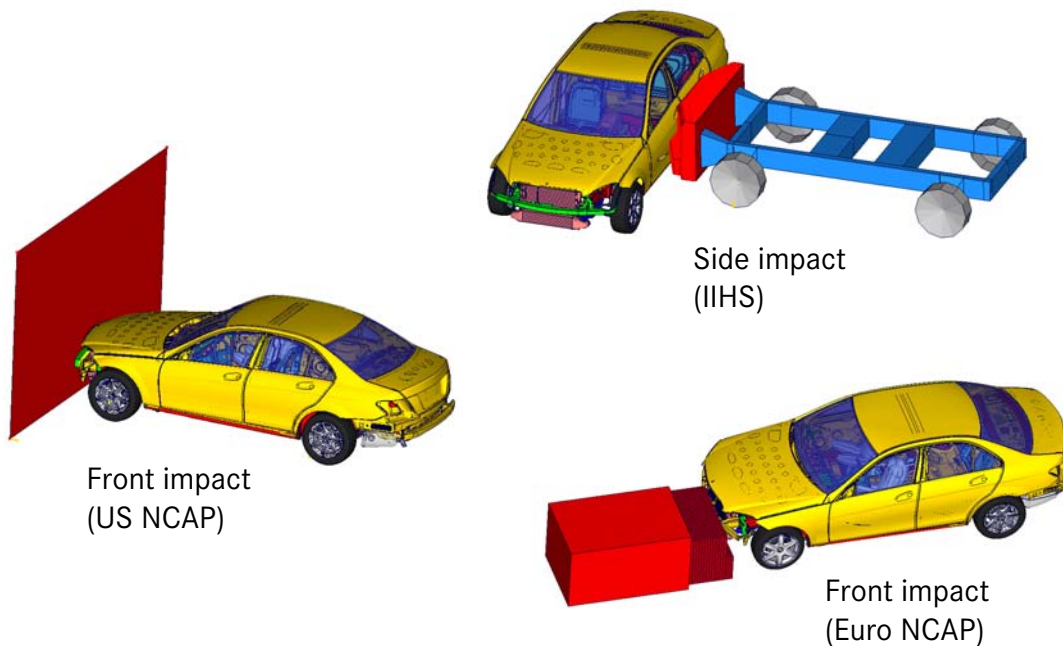
dynamic loading



# Verification & validation process Full car crash simulation validation



## Full car crash simulation



## Summary

- Comprehensive investigations to find a suitable model for different adhesives in crashworthiness simulations has been performed
- Adhesives must be categorized (rubber, foam, structural adhesive bonding) with respect to the application first
- For structural adhesive bonding, most of the shown visco-plastic models are able to describe the material behaviour.
- Verification process must contain peeling tests for correct representation of energy dissipation
- Continuum formulations are preferred for detailed modelling (correct mechanical behavior)
- Cohesive models have advantages for substitute modelling (direct parameter fitting, independent constant thickness definition)
- Time step is not critical for substitution models, cohesive formulations have the advantage of independent thickness definition
- Full car crash simulations are still problematic in front impact load cases
- A rigorous verification and validation procedure is needed to gain robust, reliable and predictable results

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## FOSTA P676

**FOSTA-Projekt P676:**  
**Methodenentwicklung zur Berechnung von höherfesten**  
**Stahlklebverbindungen des Fahrzeugbaus unter**  
**Crashbelastung**

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Thank you for your attention!

